

Considerations about the choice of layerwise and through-thickness global functions of 3-D physically-based zig-zag theories

*Original*

Considerations about the choice of layerwise and through-thickness global functions of 3-D physically-based zig-zag theories / Icardi, U.; Urraci, A.. - In: COMPOSITE STRUCTURES. - ISSN 0263-8223. - STAMPA. - 244:(2020). [10.1016/j.compstruct.2020.112233]

*Availability:*

This version is available at: 11583/2822437 since: 2020-05-12T12:27:29Z

*Publisher:*

Elsevier Ltd

*Published*

DOI:10.1016/j.compstruct.2020.112233

*Terms of use:*

This article is made available under terms and conditions as specified in the corresponding bibliographic description in the repository

*Publisher copyright*

Elsevier postprint/Author's Accepted Manuscript

© 2020. This manuscript version is made available under the CC-BY-NC-ND 4.0 license  
<http://creativecommons.org/licenses/by-nc-nd/4.0/>. The final authenticated version is available online at:  
<http://dx.doi.org/10.1016/j.compstruct.2020.112233>

(Article begins on next page)

# Considerations about the choice of layerwise and through-thickness global functions of 3-D physically-based zig-zag theories

## Abstract

A generalization of physically-based fixed degrees of freedom 3-D zig-zag theories is developed, which allows for any arbitrary choice of layerwise and representation functions. Thereby users can choose arbitrarily the representation case-by-case. This paper aims to prove that the choice of global and layerwise functions is immaterial whenever coefficients are recalculated exactly (via symbolic calculus) by the enforcement of interfacial stress continuity, boundary conditions and equilibrium in point form, as prescribed by the elasticity theory. Vice versa, accuracy of theories partially fulfilling constraints will prove to largely depend on the assumptions made and to be inadequate when strong layerwise effects rise, under distributed/localized step loading and boundary conditions other than simply supported edges (tests carried out in closed-form). The present and previous authors' theories are tested with the aim to understand in which cases an adequate level of accuracy is still achieved by lower-order theories that are derived as particularizations.

## Keywords

Physically-based zig-zag theories generalized formulation, Arbitrary through-thickness representation of global and layerwise functions with fixed d.o.f., Transverse shear and normal stresses, Laminated and soft-core sandwich plates and beams, Layers stiffness ratio, Analytical solutions

## 1. Introduction

As well known, laminated fibre-reinforced and sandwich composites offer excellent specific strength and stiffness, fatigue and energy absorption properties, better resistance to corrosion than metals and greater design flexibility. To ward off any possible catastrophic failure or intolerable loss of performance of these materials, an accurate prediction of through-thickness displacement, strain and stress fields is crucial [1]. Displacements have to be  $C^0$  continuous at interfaces and with the suited slope (zig-zag effect) so that out-of-plane stresses are continuous and satisfy local equilibrium equations.

So far many laminated plate and shell theories with varying degrees of accuracy, computational costs and memory storage occupation have been developed, as explained in detail in the book by Reddy [2] and in the papers by Reddy and co-workers [1], [3], Vasilive and Lur'e [4], Lur'e, and Shumova [5], Noor et al. [6], Altenbach [7], Carrera [8-12], Qatu [13], Qatu et al. [14], Wanji and Zhen [15], Khandan et al. [16] and Kapuria and Nath [17].

Equivalent single-layer [18]-[21], discrete-layer [22], [23], zig-zag [11], [24], hierarchical [25], [26] and axiomatic/asymptotic [27], [28] theories can be categorized, which further subdivide into displacement-based and mixed formulations (displacements, strains and stress fields chosen separately).

Equivalent single layer theories are of limited validity, not even being able to predict overall behaviour quantities, but even this limited goal could be disregarded, as shown among many others by Icardi and Sola [29], Icardi and Urraci [30,31], Kapuria et al. [32], Zhen and Wanji [33], Burlayenko et al. [34] and Jun et al. [35] for cases with a strong transverse anisotropy. A forcibly schematic description like that of this article leads to summarize discrete-layer theories as potentially accurate irrespective of lay-up, property variation across the thickness, loading and boundary conditions, but also problematic because they could overwhelm the computational capacity when structures of industrial interest are analyzed, owing to their too much number of variables. Recent accurate, versatile and efficient theories like Carrera's unified formulation [8] and refined zig-zag theories [29-31,36] are instead suitable for analysis of industrial structures. Theory [8] allows displacements to take arbitrary forms that can be chosen by the user as an input of the analysis and it is able to get existing theories as particularizations. This paper intends to pursue the same purpose with a new generalized zig-zag theory.

Zig-zag theories subdivide into: (i) Di Sciuva's like physically-based theories (see [37-39] as examples), or (ii) Murakami's like kinematic-based mixed theories, (see, e.g. [11],[40], [41]), given the different type of zig-zag functions used. Both strike the right balance between accuracy and cost saving, allowing designers' demand of theories in a simple already accurate form to be met.

Theories (i) generally have a fixed number of unknowns irrespective of the number of constituent layers. Their layerwise contributions are the product of linear [37] (or nonlinear [42]) zig-zag functions and unknown zig-zag amplitudes, inferred by enforcing the fulfilment of stress continuity conditions at layer interfaces.

Theories (ii) assume zig-zag functions featuring a periodic change of sign of the slope of displacements at interfaces, as occurring for periodic laminations, therefore without taking into account orientation angle, material properties and thickness of constituent layers. Their number of variables (kinematic and stress quantities) depends upon the number of constituent/computational layers. They can be inaccurate [30, 31, 43, 44], but they enable a more easily obtainment of  $C^0$  formulations of plate theories for the development of efficient finite elements. Mixed multilayered theories accurately predict stresses, with the merit of keeping simpler kinematics as is evident from the papers by Tessler et al. [36], Kim and Cho [45], Barut et al. [46], Iurlaro et al. [47] and Zhen and Wanji [48]. When strong layerwise effects rise, (ii) require many more degrees of freedom and/or a higher expansion order than (i), as it can be seen, for example, comparing the results of [30] and [49].

Efficient and accurate theories where zig-zag functions are not explicitly incorporated based on a global-local superposition of displacement fields were proposed by Li and Liu [50] and refined by Zhen and Wanji, e.g. [51], which in the version of Shariyat [52] consider a non-uniform transverse displacement across the thickness.

Theories with a hierarchical set of locally defined polynomials where neither zig-zag contributions are incorporated in the kinematics, nor post-processing steps are required (see, [25], [26], Catapano et al. [53])

and de Miguel et al. [54]) have been developed, for favouring numerical efficiency. Anyway, they require a quite large number of variables to mitigate the effects of the omitted enforcement of interfacial stress contact conditions, so to avoid small interfacial stress jumps.

Physically-based 3D zig-zag theories like [29-31] also allow a variable representation arbitrarily chosen by the user, without increasing their only 5 d.o.f. , while Strain Energy Updating Technique [55] can be used to obtain a  $C^0$  formulation. Because their coefficients are redefined across the thickness, they can be counted among those with variable-kinematics (see, e.g. Vescovini and Dozio [56]).

So far, power series expansion, hierarchic polynomials, Taylor's series, trigonometric and exponential functions, a combination of both and radial basis functions (see [57] to [66]) have been used to represent variables across the thickness and a high sensitivity to the different assumptions made was shown.

Going into more detail regarding physically-based 3D zig-zag theories, in Icardi and Sola [29] a displacement-based laminated plate theory with piecewise cubic in-plane displacements, a fourth-order transverse displacement and only 5 d.o.f. (mid-plane displacements and rotations), referred as ZZA, was developed, whose coefficients are redefined across the thickness by imposing the fulfilment of elasticity theory constraints (interfacial stress compatibility conditions, stress boundary conditions and local equilibrium equations at arbitrary selected points across the thickness).

Variants of ZZA were developed in [30], assuming different layerwise functions, in order to evaluate how the different choices affect accuracy. For the same purpose, different forms of representation of global functions and various layerwise functions (as well as without them) were assumed in [67]-[71]. In an attempt of lowering the computational cost, a mixed variant of the already very efficient ZZA theory was developed in [31] within the framework of Hu-Washizu variational theorem, so to retain separately only the essential contributions of displacement, strain and stress fields. Theories [29,30,67-71], all developed by using symbolic calculus to exactly fulfil the elasticity theory constraints, were proven to be efficient and suitable to analyse challenging elastostatic and dynamic cases, including pumping modes which are usually solved through FEM. In effects, their accuracy was proven to be similar to that of discrete layer and layerwise models with a very high through-thickness expansion order of variables and so a larger computational burden.

A not negligible advantage of these theories is that there is no need to approximate loading with a series expansion, because its mathematical expression is used to exactly compute the work of external forces via symbolic calculus. Another advantage is that also in-plane displacement and stress continuity can be enforced, with the aim to analyse also structures with step variable properties along in-plane directions and not only across the thickness. Moreover, stresses of theories [29,30,67-71] are obtained from constitutive relations and there is no need to post-processing.

Summing up the features of theories [29,30,67-71], similarly to [50] - [52], it is not necessary to include specific zigzag functions to respect the interfacial out-of-plane stress compatibility, because a number of coefficients can be redefined for each layer for this purpose. However, unlike [50] - [52] and all conventional plate theories, in [29,30,67-71] all coefficients can be redefined across the thickness so that stress-boundary conditions are met at the outer layers and equilibrium can be satisfied in a strong point form within inner layers. Similarly to hierarchical and asymptotic theories, the representation can be

enriched in order to achieve a better accuracy, but the expansion order and the number of variables do not grow once the piecewise cubic-quartic representation adopted is fragmented into computational layers and/or changing the type of representation from layer to layer and differently for each displacement.

Unlike the theories that today are very popular for their accuracy and versatility (e.g. [8] and subsequent developments), [29,30,67-71] are formulated by respecting physical constraints exactly (from which the appellation of physically-based theories follows) so as to limit the computational burden, rather respecting them in a limit sense increasing the expansion order and / or the number of variables.

A generalization of previous 3-D zig-zag theories is developed in this paper, whose formulation allows user to arbitrarily choose layerwise and representation functions, also differently for each displacement and for each layer. Nevertheless any number of d.o.f. could be assumed (not necessarily just mid-plane displacements and rotations), it is here limited to five to carry out comparisons with ZZA and other previous theories under the same conditions. From the present generalized theory an approximate non-plate theory (number of d.o.f. not fixed that depends from the expansion order imposed) AT-3D is derived, which can be used as reference solution when exact one is unavailable. Given its generality, also hierarchical theories could be obtained by the present theory, which however won't be considered in this research.

Six theories different from those of [29]-[31] and [67]-[71] are particularized in (3.1) assuming sinusoidal, exponential, power series or a combination of them (which can be different for each displacement and from layer to layer) to describe the variation of displacements across the thickness.

Fourteen additional lower order theories are developed in (3.2)-(3.8) to test along with the former six if a plate theory with fixed d.o.f. can enable a kinematic-variable representation across the thickness requiring a lower computational burden than existing theories with same features, and to understand in which cases it is still allowed to reach an adequate level of accuracy. The results will show that a piecewise cubic and a fourth-order polynomial for in-plane and transverse displacements respectively achieve this goal. In one theory, equilibrium equations are enforced in weak form, instead than in a point form as for all others theories mentioned, in order to verify the degree of accuracy achievable. Results will confirm on a broader basis what preliminarily demonstrated in [29]-[31] and [67]-[71], that the choice of zig-zag and representation functions is immaterial if coefficients are redefined across the thickness and calculated by imposing full set of physical constraints.

Table 1 provides a quick reference pattern of all cases considered in the numerical applications and lists details on length-to-thickness ratio, lay-up, loading and boundary conditions. Mechanical properties of materials are in Table 2, Table 3 reports trial functions, expansion order and equilibrium points position across the thickness for each case, Table 4 contains normalizations, Table 5 records a brief description of theories, while the processing time is listed in Table 6.

## 2. Theoretical Framework

Notations and basic assumptions, that are common to all the theories, are discussed in the following section. Features of the new theories introduced in this paper are examined in details. Readers are referred to the literature quoted for previously developed theories.

### 2.1 Basic assumptions and main notations

This study is restricted to multilayered elastic plates subject to small deformations, as representative of laminated and sandwiches. Loading is assumed conservative, strains are assumed to be infinitesimal, while constituent layers are assumed to be linearly elastic, to have orthotropic properties, a uniform arbitrary thickness  $h^k$  and to be perfectly bonded to each other (bonding resin interlayer is disregarded).

A rectangular, right-handed Cartesian coordinate reference system  $\Omega$  on the middle reference plane is assumed as the reference frame. As easily allowed by symbolic calculus used to develop theories,  $\Omega$  could be assumed differently to any other position, so to prevent an eventual zeroing of coefficients for

certain lay-ups and theories. The thickness coordinate is  $\zeta$  ( $\zeta \in \left[-\frac{h}{2}, \frac{h}{2}\right]$ ,  $h$  being the overall

thickness) and a comma is used to indicate spatial derivatives, e.g.  $(\cdot)_{,\alpha} = \partial(\cdot)/\partial\alpha$ .  $L\alpha$  and  $L\beta$

symbolize the plate side-length in the  $\alpha$ - and  $\beta$ -directions. Strains and stresses are indicated respectively as  $\varepsilon_{ij}$ ,  $\sigma_{ij}$  ( $i, j = 1, 2, 3$ ;  $1 \equiv \alpha, 2 \equiv \beta, 3 \equiv \zeta$ ). The upper and lower positions of layer interfaces are

indicated with  ${}^{(k)}\zeta^+$  and  ${}^{(k)}\zeta^-$ , respectively, the superscript (or in other cases the subscript)  $k$  being

used to indicate that a quantity belongs to the layer  $k$ . To indicate that a quantity is evaluated on upper and lower faces of the laminate, the markers  $u$  and  $l$  are used, respectively.

Except for a couple of theories for which the appropriate specifications will be given below, all remaining theories have the following five d.o.f.  $u_\alpha^0$ ,  $u_\beta^0$ ,  $w^0$ ,  $\Gamma_\alpha^0$ ,  $\Gamma_\beta^0$ , consisting of middle plane displacement components and shear rotations of the normal

### 2.2 Solutions search

Solution is searched in a closed form within the framework of Rayleigh-Ritz method, in conjunction with Lagrange multipliers method. Accordingly, d.o.f. are expressed as a truncated series expansion of unknown amplitudes and trial functions that individually satisfy the prescribed boundary conditions. The trial functions are explicitly defined in Table 3 for each specific case along with the expansion order and normalizations used. The same trial functions and expansion order are shared by all theories, in order to compare them under the same conditions. The methodology to satisfy the mechanical boundary conditions (transverse shear stress resultant force equals the constraint force and resultant couple of in-plane stresses, where they are not identically satisfied) based on Lagrange multipliers method is the same of [31], where readers can find all details here omitted. The derivation of governing equations will be

omitted to contain the paper length because they are obtainable in a straightforward way with standard techniques.

### 2.3 ZZA displacement-based theory

Hereafter the main features of zig-zag adaptive theory (ZZA [29]) are summarized, it being the basis from which all theories developed and assessed in this paper evolved. The following through-thickness displacement field is postulated:

$$\begin{aligned}
u_\alpha(\alpha, \beta, \zeta) &= \left[ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) \right]_0 + \left[ F_\alpha^u(\alpha, \beta, \zeta) \right]_i + \\
&\quad \left[ \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta)(\zeta - \zeta_k) H_k(z) + \sum_{j=1}^{n_\zeta} C_\alpha^j(\alpha, \beta) H_j(\zeta) \right]_c \\
u_\zeta(\alpha, \beta, \zeta) &= \left[ w^0(\alpha, \beta) \right]_0 + \left[ F^\zeta(\alpha, \beta, \zeta) \right]_i + \left[ \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta)(\zeta - \zeta_k) H_k(\zeta) + \right. \\
&\quad \left. \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta)(\zeta - \zeta_k)^2 H_k(\zeta) + \sum_{j=1}^{n_\zeta} C_\zeta^j(\alpha, \beta) H_j(\zeta) \right]_c
\end{aligned} \tag{1}$$

Symbols  $n_i$  and  $n_\zeta$  are used to distinguish the number of physical interfaces from that of mathematical layer interfaces, respectively.

It could be noticed that linear- $[\dots]_0$ , higher- $[\dots]_i$  and layerwise- $[\dots]_c$  contributions are incorporated, whose purpose is defined as follows. The first term  $[\dots]_0$  introduces the functional degrees of freedom, while the second one  $[\dots]_i$  contains higher-order contributions, while the third contribution is characteristic of physically-based zig-zag theories.

Although  $\left[ F_\alpha^u(\alpha, \beta, \zeta) \right]_i$ ,  $\left[ F^\zeta(\alpha, \beta, \zeta) \right]_i$  could be chosen entirely arbitrarily, the following power series expansion is chosen:

$$\begin{aligned}
\left[ F_\alpha^u(\alpha, \beta, \zeta) \right]_i &= \left[ C_\alpha^i(\alpha, \beta)\zeta^2 + D_\alpha^i(\alpha, \beta)\zeta^3 + (O\zeta^4 \dots) \right]_i = \left[ {}_3(\tilde{\cdot})_\alpha \right]_i + \left[ (O\zeta^4 \dots) \right]_i = \\
&= \left[ A_{\alpha 2}(\alpha, \beta)\zeta^2 + A_{\alpha 3}(\alpha, \beta)\zeta^3 \right] + A_{\alpha 4}(\alpha, \beta)\zeta^4 + \dots + A_{\alpha n}(\alpha, \beta)\zeta^n
\end{aligned} \tag{2}$$

$$\begin{aligned}
\left[ F^\zeta(\alpha, \beta, \zeta) \right]_i &= \left[ b^i(\alpha, \beta)\zeta + c^i(\alpha, \beta)\zeta^2 + d^i(\alpha, \beta)\zeta^3 + e^i(\alpha, \beta)\zeta^4 + (O\zeta^5 \dots) \right]_i = \\
&= \left[ {}_4(\tilde{\cdot})_\zeta \right]_i + \left[ (O\zeta^5 \dots) \right]_i = \left[ A_{\zeta 1}(\alpha, \beta)\zeta + A_{\zeta 2}(\alpha, \beta)\zeta^2 + A_{\zeta 3}(\alpha, \beta)\zeta^3 + \right. \\
&\quad \left. + A_{\zeta 4}(\alpha, \beta)\zeta^4 \right] + A_{\zeta 5}(\alpha, \beta)\zeta^5 + \dots + A_{\zeta n}(\alpha, \beta)\zeta^n
\end{aligned}$$

so to include theory [42] as a particularization of ZZA. Contributions  $\left[ (O\zeta^4 \dots) \right]_i$ ,  $\left[ (O\zeta^5 \dots) \right]_i$  are characteristic of ZZA, while  $\left[ {}_3(\tilde{\cdot})_\alpha \right]_i$ ,  $\left[ {}_4(\tilde{\cdot})_\zeta \right]_i$  are the same as in [42]. Note that functions  $\left[ F_\alpha^u \right]_i$ ,  $\left[ F^\zeta \right]_i$  aren't just depending on  $\zeta$  because apexes and subscripts  ${}_\alpha^u$ ,  ${}_\zeta^\zeta$  represent the functional dependence on the d.o.f. that are function of in-plane coordinates. Expressions of coefficients  $C_\alpha^i$ ,  $D_\alpha^i$ ,  $b^i$  to  $e^i$  are obtained by enforcing the fulfilment of stress boundary conditions:

$$\sigma_{\alpha\zeta} = \sigma_{\zeta\zeta,\zeta} = 0; \quad \sigma_{\zeta\zeta} = p^0(\pm) \quad (3)$$

$p^0(\pm)$  being the distributed loading acting on at the upper (+) and lower (-) boundingfaces, and of local equilibrium equations:

$$\sigma_{\alpha\beta,\beta} + \sigma_{\alpha\zeta,\zeta} = b_\alpha; \quad \sigma_{\alpha\zeta,\alpha} + \sigma_{\zeta\zeta,\zeta} = b_\zeta \quad (4)$$

at points across the thickness, positioned where residual are the largest. The through-thickness redefinition of coefficients by (3) and (4) determines the adaptive appellation attributed to ZZA theory. Applied distributed loadings are managed via symbolic calculus like general (continuous or discontinuous) functions  $\chi(\alpha, \beta)$  acting at upper and/or lower faces, so it can be avoided to express them as a series expression that could limit accuracy.

Layerwise contributions  $[\dots]_c$  allow a priori fulfilment of stress compatibility conditions at physical and mathematical layer interfaces, so  $\Phi_\alpha^k, \Psi^k, \Omega^k$  are determined by imposing

$$\sigma_{\alpha\zeta}^{(k)z^+} = \sigma_{\alpha\zeta}^{(k)z^-}; \quad \sigma_{\zeta\zeta}^{(k)z^+} = \sigma_{\zeta\zeta}^{(k)z^-}; \quad \sigma_{\zeta\zeta,\zeta}^{(k)z^+} = \sigma_{\zeta\zeta,\zeta}^{(k)z^-} \quad (5)$$

Here - and + indicate the position just before and just after the interface, respectively). Contributions  ${}_\alpha C_u^j H_j$  and  $C_\zeta^j H_j$  in (1) restore the continuity of displacements:

$$u_\alpha^{(k)z^+} = u_\alpha^{(k)z^-}; \quad u_\zeta^{(k)z^+} = u_\zeta^{(k)z^-} \quad (6)$$

since (2) which is contained within  $[\dots]_i$  can be assumed differently from point to point across the thickness. At the end of this brief discussion it is reminded that symbolic calculus enables that expressions of coefficients and zig-zag amplitudes to be obtained in exact form and once and for all. It is also reminded that just a third/fourth order representation is required to obtain accurate results [30]. In order to assess whether and when the choice of zig-zag functions can be immaterial, in the numerical applications a new variant of ZZA called ZZA\_MHR is considered, wherein Murakami's zig-zag function multiplied for amplitudes that are recalculated across the thickness are assumed as the layerwise functions, instead of  $[\dots]_c$  by ZZA.

#### 2.4 Previously developed theories that are considered for sake of comparisons

They comprise theories HWZZ [31], HWZZM [30], MHR [31], ZZA\* [30] and ZZ [42], whose features are briefly summarized as follows. HWZZ is a mixed HW theory<sup>3,4</sup>, neglecting quadratic zig-zag and higher-order adaptive contribution of (1) from displacement field, out-of-plane strains are the same of (1), while out-of-plane stresses by integrating local equilibrium equation. HWZZM has the same features of HWZZ, except zig-zag functions which are modified versions of Murakami's layerwise functions with amplitudes that are redefined by enforcing (5).

MHR assumes a piecewise cubic in-plane displacement including original Murakami's layerwise functions, namely amplitudes are not redefined across the thickness, and a fourth-order polynomial transverse displacement. ZZA\* has the same features of ZZA, except zig-zag functions are replaced by



appropriate additional contributions whose amplitudes are redefined across the thickness. ZZ is based on a piecewise cubic in-plane representation a piecewise fourth-order for transverse displacement, the same as (1), except that [...] are not redefined across the thickness.

### 3. New theories of this paper

New theories, which constitutes the main theoretical contribution of this paper, are proposed as a generalization of ZZA and all previously developed theories by the authors [29-31, 42, 55, 68-71], in order to prove the objective set in the introductory section. Initially, the displacement field is thought in the following form:

$$\begin{aligned} u_\alpha^j &= \sum_{i=0}^{n_\alpha=3} \left[ {}^j C_\alpha^i(\alpha, \beta) F_\alpha^i(\zeta) \right] \\ u_\zeta^j &= \sum_{i=0}^{n_\zeta=4} \left[ {}^j C_\zeta^i(\alpha, \beta) G^i(\zeta) \right] \end{aligned} \quad (7)$$

that does not contain zig-zag functions because coefficients are redefined across the thickness, being calculated by imposing the full set of physical constraints (3)-(6) in order to get accurate results using a low order of expansion. Subsequently, particularizations are developed from (7) in the next section (3.1).

Symbol  $j$  represents the mathematical layers, so,  ${}^j C_\alpha^i$ ,  ${}^j C_\zeta^i$ ,  $F_\alpha^i(\zeta)$  and  $G^i(\zeta)$  can be represented differently across the thickness. Coefficients  ${}^1 C_\alpha^0$ ,  ${}^1 C_\alpha^1$  and  ${}^1 C_\zeta^0$  are assumed as d.o.f. To obtain (1) as a particularization of (7) it is necessary that  ${}^1 C_\alpha^0 = u_\alpha^0$ ,  ${}^1 C_\alpha^1 = \Gamma_\alpha^0 - w_{,\alpha}^0$  and  ${}^1 C_\zeta^0 = w^0$  (

$F^0(\zeta) = G^0(\zeta) = 1$ ,  $F^1(\zeta) = \zeta$ ), while the remaining terms are calculated by enforcing physical

constraints (3)-(6). Of course, the expansion order  $n_\alpha$  and  $n_\zeta$ , which reflects in the number of unknowns from one interface (physical or computational) to another, must be chosen in accordance with the physical constraints to be imposed. Once the interface has passed, the coefficients are redefined and then it is possible that the representation changes. Functions  $F^i(\zeta)$  and  $G^i(\zeta)$  can be freely chosen by the user (symbolic calculations being performed automatically regardless of the choices made).

In numerical applications, theory AT-3D with  $F^i(\zeta) = \zeta^i$  and  $G^i(\zeta) = \zeta^i$  is considered, wherein  $n_\alpha$  and  $n_\zeta$  can be arbitrarily assumed, but anyway so that  ${}^j C_\alpha^i$  and  ${}^j C_\zeta^i$  are in number greater than physical constraints (3)-(6). In the form described in this section, (7) constitutes a 3-D theory which owing to its generality can be used as a surrogate of the exact solution when it is not available [71].

#### 3.1 Particularizations with $u_\alpha^0$ , $\Gamma_\alpha^0 - w_{,\alpha}^0$ , $w^0$ as d.o.f.

Particularizations of (7) having same d.o.f. of ZZA are developed in this section assuming  $F^i(\zeta)$  and  $G^i(\zeta)$  as a mixture of trigonometric and exponential functions randomly selected for each theory, to

demonstrate that their choice is immaterial whenever (3)-(6) are enforced and exact relations are found from such enforcement via symbolic calculus.

Theory name	Function	thickness range
<b>NOZZG</b>	$F_{\alpha}^i(\zeta) = G^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ e^{(i \zeta / h_j)} & \text{for } i > 1 \end{cases}$	$(-h/2 \leq \zeta \leq h/2)$ (8a)
<b>ZZA_PP34</b>	$F_{\alpha}^i(\zeta) = G^i(\zeta) = (\zeta)^i$	$(-h/2 \leq \zeta \leq h/2)$ (8b)
<b>ZZA_PT34</b>	$F_{\alpha}^i(\zeta) = G^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ \cos(i\pi\zeta / 2h) & \text{for } i = 2, 4 \\ \sin((i+1)\pi\zeta / 2h) & \text{for } i = 3 \end{cases}$	$(-h/2 \leq \zeta \leq h/2)$ (8c)
<b>ZZA_PM34</b>	$F_{\alpha}^i(\zeta) = G^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ \exp(\zeta / h) & \text{for } i = 2 \\ \sin(\pi\zeta / 2h) & \text{for } i = 3 \\ \cos(\pi\zeta / 2h) & \text{for } i = 4 \end{cases}$	$(-h/2 \leq \zeta \leq h/2)$ (8d)
<b>ZZA_PMTP34</b>	$F_{\alpha}^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ \exp(\zeta / h) & \text{for } i = 2 \\ \sin(\pi\zeta / 2h) & \text{for } i = 3 \\ \cos(\pi\zeta / 2h) & \text{for } i = 4 \end{cases}$	$(-h/2 \leq \zeta \leq h/2)$
	$F_{\beta}^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ \cos(i\pi\zeta / 2h) & \text{for } i = 2, 4 \\ \sin((i+1)\pi\zeta / 2h) & \text{for } i = 3 \end{cases}$	$(-h/2 \leq \zeta \leq h/2)$ (8e)
	$G^i(\zeta) = (\zeta)^i$	$(-h/2 \leq \zeta \leq h/2)$
	$F_{\alpha}^i(\zeta) = G^i(\zeta) = (\zeta)^i$	$(-h/2 \leq \zeta \leq -0.45h)$
<b>ZZA_PPM34</b>	$F_{\alpha}^i(\zeta) = G^i(\zeta) = \begin{cases} 1 & \text{for } i = 0 \\ \zeta & \text{for } i = 1 \\ \exp(\zeta / h) & \text{for } i = 2 \\ \sin(\pi\zeta / 2h) & \text{for } i = 3 \\ \cos(\pi\zeta / 2h) & \text{for } i = 4 \end{cases}$	$(-0.45h < \zeta \leq 0.4h)$ (8f)
	$F_{\alpha}^i(\zeta) = G^i(\zeta) = (\zeta)^i$	$(0.4h < \zeta \leq h/2)$

Theories that only partially satisfy physical constraints are set hereafter, which are considered for the purpose of comparisons.

### 3.2 ZZAM\_P3P4 theory

It has the following representation form:

$$\begin{aligned}
u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + [ \sum_{k=1}^3 C_{k-\alpha}^i(\alpha, \beta) F_k^\alpha(\zeta) + C_\alpha^i ]_{i+c} \\
u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + [ \sum_{k=1}^4 D_k^i(\alpha, \beta) G_k(\zeta) + C_\zeta^i ]_{i+c} \\
C_{k-\alpha}^{i=1} &= 0; \quad C_\alpha^{i=1} = 0; \quad C_\zeta^{i=1} = 0; \quad F_k^\alpha(\zeta) = G_k(\zeta) = (\zeta)^k
\end{aligned} \tag{9}$$

It does not contain zig-zag contributions, but still uses a power function representation and coefficients are still redefined across the thickness through the enforcement of (3)-(6). The substantial difference with respect to all previous theories is that equilibrium conditions are imposed in integral form:

$$\int_{\zeta_i^-}^{\zeta_i^+} (\sigma_{\alpha\beta, \beta} + \sigma_{\alpha\zeta, \zeta}) \Psi_\alpha d\zeta = \int_{\zeta_i^-}^{\zeta_i^+} (\sigma_{\alpha\zeta, \alpha} + \sigma_{\zeta\zeta, \zeta}) \Psi_\zeta d\zeta \tag{10}$$

so, is no longer punctual, that is computationally more advantageous.

### 3.3 Theory PP23

This theory assume a lower-order of expansion, because less equilibrium points are considered (in-plane displacements are parabolic, while the transverse one is cubic). For the rest, coefficients are redefined across the thickness by enforcing (3)-(6).

$$\begin{aligned}
u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + [ \sum_{k=1}^2 C_{k-\alpha}^i(\alpha, \beta) F_k^\alpha(\zeta) + C_\alpha^i ]_{i+c} \\
u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + [ \sum_{k=1}^3 D_k^i(\alpha, \beta) G_k(\zeta) + C_\zeta^i ]_{i+c} \\
C_{k-\alpha}^{i=1} &= 0; \quad C_\alpha^{i=1} = 0; \quad C_\zeta^{i=1} = 0; \quad F_k^\alpha(\zeta) = G_k(\zeta) = (\zeta)^k
\end{aligned} \tag{11}$$

### 3.4 ZS1, ZS1\_1, ZS1\_2, ZS1\_3 and ZS1\_4

For these theories with a partial fulfilment of (3)-(6), various contributions are cut off, in order to highlight whether they are still accurate.

- ZS1.

$$\begin{aligned}
u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta) (\zeta - \zeta_k) H_k(\zeta) \\
u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta) (\zeta - \zeta_k) H_k(\zeta) + \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta) (\zeta - \zeta_k)^2 H_k(\zeta)
\end{aligned} \tag{12}$$

Zig-zag amplitudes  $\Phi_\alpha^k$ ,  $\Psi^k$  and  $\Omega^k$  are calculated by imposing the fulfilment of stresses compatibility (5) at interfaces Therefore the only redefined coefficients are zig-zag amplitudes.

- ZS1\_1. Layerwise functions of (12) are substituted with  $C_\alpha^i(\alpha, \beta)\zeta$ ,  $\Psi^i(\alpha, \beta)\zeta$  and  $\Omega^i(\alpha, \beta)\zeta^2$ ,

- ZS1\_2. It is developed as a mixed variant of ZS1 wherein the same displacements are assumed but out-of-plane stresses are assumed apart by integrating local equilibrium equations, within the framework of HR variational theorem.

- ZS1\_3. Is a mixed HR theory derived from ZS1 assuming the same displacement field, while out-of-plane stresses are assumed apart being those coming from ZZAM\_P3P4.

- ZS1\_4. Considers the same displacement field of ZS1\_1 but different  $\Gamma_\alpha^0$  d.o.f. are used, that are referred to the top face instead of middle reference plane:

$$\begin{aligned} u_\alpha(\alpha, \beta, \zeta) &= \left[ u_\alpha^0(\alpha, \beta) + (\zeta - h/2)\Gamma_\alpha^0(\alpha, \beta) - \zeta w^0(\alpha, \beta)_{,\alpha} \right]_0 + C_\alpha^i(\alpha, \beta)\zeta + D_\alpha^i \\ u_\zeta(\alpha, \beta, \zeta) &= \left[ w^0(\alpha, \beta) \right]_0 + \Psi^i(\alpha, \beta)\zeta + \Omega^i(\alpha, \beta)\zeta^2 + C_\zeta^i \quad (C_\alpha^{i+1} = 0) \end{aligned} \quad (13)$$

### 3.5 Theory ZS2

Displacement field is:

$$\begin{aligned} u_\alpha(\alpha, \beta, \zeta) &= \left[ u_\alpha^i(\alpha, \beta) L_1(\zeta) + u_\alpha^{i+1}(\alpha, \beta) L_2(\zeta) \right]_0 \\ u_\zeta(\alpha, \beta, \zeta) &= \left[ w^i(\alpha, \beta) L_1(\zeta) + w^{i+1}(\alpha, \beta) L_2(\zeta) \right]_0 + \Omega^k(\alpha, \beta)\zeta^2 \end{aligned} \quad (14)$$

L1 and L2 being linear Lagrange polynomials

$$L_1(\zeta) = 1 - \frac{\zeta - \zeta_b^i}{\zeta_t^i - \zeta_b^i}; \quad L_2(\zeta) = \frac{\zeta - \zeta_b^i}{\zeta_t^i - \zeta_b^i} \quad (15)$$

The suffix b and t are used to indicate upper and lower coordinates of each generic lamina i and displacements at upper and lower bounding faces ( $u_\alpha^{top} = u_\alpha^{n_i}$ ,  $u_\alpha^{bottom} = u_\alpha^1$ ,  $w^{top} = w^{n_i}$ ,  $w^{bottom} = w^1$ ,  $n_i$  number of computational layers) are assumed as the functional d.o.f. So, other coefficients  $u_\alpha^i$  and  $w^i$  are obtained as functions of  $u_\alpha^{top}$ ,  $u_\alpha^{bottom}$ ,  $w^{top}$ ,  $w^{bottom}$  by imposing (5).

### 3.6 Theory ZS3

The displacement field is structured as follows:

$$\begin{aligned} u_\alpha(\alpha, \beta, \zeta) &= \left[ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) \right]_0 + C_\alpha^i(\alpha, \beta)\zeta + D_\alpha^i(\alpha, \beta)\zeta^2 + C_{\alpha-u}^i \\ u_\zeta(\alpha, \beta, \zeta) &= \left[ w^0(\alpha, \beta) \right]_0 \end{aligned} \quad (16)$$

Terms  $C_\alpha^i$   $D_\alpha^i$  are obtained by imposing (3)-(6), as well as the boundary condition on transverse shear

$$T = \int_{-h/2}^{h/2} \sigma_{\alpha\zeta} d\zeta, \text{ as considered in certain theories in literature while, } C_{\alpha-u}^i \text{ by imposing (6).}$$

### 3.7 Theories ZS3\_1 and ZS3\_2

Theory ZS3\_1 assumes a piecewise parabolic representation for all displacements, free of zig-zag contributions, whose coefficients are calculated even imposing (5)-(6):

$$\begin{aligned} u_\alpha(\alpha, \beta, \zeta) &= \left[ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) \right]_0 + C_\alpha^i(\alpha, \beta)\zeta + D_\alpha^i(\alpha, \beta)\zeta^2 + C_{\alpha-u}^i \\ u_\zeta(\alpha, \beta, \zeta) &= \left[ w^0(\alpha, \beta) \right]_0 + \Psi^i(\alpha, \beta)\zeta + \Omega^i(\alpha, \beta)\zeta^2 + C_\zeta^i \end{aligned} \quad (17)$$

Theory ZS3\_2 is developed assuming the same displacements field of ZS3\_1 but out-of-plane stresses are assumed apart by integrating local equilibrium equations.

### 3.8 Theories ZZAS1 to ZZAS4

The in-plane displacement is assumed linear while the transverse displacement is uniform for layers ( $i = i_j$ ). For  $i \neq i_j$  they are assumed as the same of (1).

$$\begin{aligned}
 u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + \\
 &+ \begin{cases} [ F_\alpha^u(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \sum_{k=1}^{n_i} C_u^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ + C_\alpha^i(\alpha, \beta)\zeta + B_\alpha^i(\alpha, \beta) & \text{if } i = i_j \end{cases} \\
 u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + \\
 &+ \begin{cases} [ F^\zeta(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \\ + \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta)(\zeta - \zeta_k)^2 H_k(\zeta) + \sum_{k=1}^{n_i} C_\zeta^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ B_\zeta^i(\alpha, \beta) & \text{if } i = i_j \end{cases} \quad (18)
 \end{aligned}$$

ZZAS2 assumes a parabolic transverse displacement for layers  $i = i_j$ , while the same in-plane displacement of ZZAS1 is maintained:

$$\begin{aligned}
 u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + \\
 &+ \begin{cases} [ F_\alpha^u(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \sum_{k=1}^{n_i} C_u^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ + C_\alpha^i(\alpha, \beta)\zeta + B_\alpha^i(\alpha, \beta) & \text{if } i = i_j \end{cases} \quad (19)
 \end{aligned}$$

$$\begin{aligned}
 u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + \\
 &+ \begin{cases} [ F^\zeta(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \\ + \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta)(\zeta - \zeta_k)^2 H_k(\zeta) + \sum_{k=1}^{n_i} C_\zeta^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ C_\zeta^i(\alpha, \beta)\zeta + D_\zeta^i(\alpha, \beta)\zeta^2 + B_\zeta^i(\alpha, \beta) & \text{if } i = i_j \end{cases}
 \end{aligned}$$

ZZAS3 assumes a parabolic-cubic displacements field for  $i_j$  layers and the same in-plane field of ZZAS1 and ZZAS2:

$$\begin{aligned}
 u_\alpha(\alpha, \beta, \zeta) &= [ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) ]_0 + \\
 &+ \begin{cases} [ F_\alpha^u(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \sum_{k=1}^{n_i} C_u^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ C_\alpha^i(\alpha, \beta)\zeta + D_\alpha^i(\alpha, \beta)\zeta^2 + B_\alpha^i(\alpha, \beta) & \text{if } i = i_j \end{cases} \quad (20)
 \end{aligned}$$

$$\begin{aligned}
 u_\zeta(\alpha, \beta, \zeta) &= [ w^0(\alpha, \beta) ]_0 + \\
 &+ \begin{cases} [ F^\zeta(\alpha, \beta, \zeta) ]_i + [ \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \\ + \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta)(\zeta - \zeta_k)^2 H_k(\zeta) + \sum_{k=1}^{n_i} C_\zeta^k(\alpha, \beta)H_k(\zeta) ]_c & \text{if } i \neq i_j \\ C_\zeta^i(\alpha, \beta)\zeta + D_\zeta^i(\alpha, \beta)\zeta^2 + E_\zeta^i(\alpha, \beta)\zeta^3 + B_\zeta^i(\alpha, \beta) & \text{if } i = i_j \end{cases}
 \end{aligned}$$

Because of limitative assumptions, theories from 3.3 to the previous one may turn out to be inaccurate. ZZAS4 is developed considering the same representation of displacements as in section 2.3 for layers  $i \neq i_j$ , while for  $i = i_j$  zig-zag functions are omitted, their sole being played by  $C_\alpha^i$  to  $B_\zeta^i$ :

$$u_\alpha(\alpha, \beta, \zeta) = \left[ u_\alpha^0(\alpha, \beta) + \zeta(\Gamma_\alpha^0(\alpha, \beta) - w^0(\alpha, \beta)_{,\alpha}) \right]_0 + \begin{cases} \left[ F_\alpha^u(\alpha, \beta, \zeta) \right]_i + \left[ \sum_{k=1}^{n_i} \Phi_\alpha^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \sum_{k=1}^{n_i} C_\alpha^k(\alpha, \beta)H_k(\zeta) \right]_c & \text{if } i \neq i_j \\ C_\alpha^i(\alpha, \beta)\zeta + D_\alpha^i(\alpha, \beta)\zeta^2 + E_\alpha^i(\alpha, \beta)\zeta^3 + B_\alpha^i(\alpha, \beta) & \text{if } i = i_j \end{cases} \quad (21)$$

$$u_\zeta(\alpha, \beta, \zeta) = \left[ w^0(\alpha, \beta) \right]_0 + \begin{cases} \left[ F^\zeta(\alpha, \beta, \zeta) \right]_i + \left[ \sum_{k=1}^{n_i} \Psi^k(\alpha, \beta)(\zeta - \zeta_k)H_k(\zeta) + \sum_{k=1}^{n_i} \Omega^k(\alpha, \beta)(\zeta - \zeta_k)^2 H_k(\zeta) + \sum_{k=1}^{n_i} C_\zeta^k(\alpha, \beta)H_k(\zeta) \right]_c & \text{if } i \neq i_j \\ C_\zeta^i(\alpha, \beta)\zeta + D_\zeta^i(\alpha, \beta)\zeta^2 + E_\zeta^i(\alpha, \beta)\zeta^3 + F_\zeta^i(\alpha, \beta)\zeta^4 + B_\zeta^i(\alpha, \beta) & \text{if } i = i_j \end{cases}$$

Unlike the previous theories, the full set of physical constraints (3)-(6) is imposed in (21), so, there is no loss of accuracy.

In conclusion, a generalization of physically-based theories has been proposed, that enables users to arbitrarily choose representation and layerwise functions, without affecting the accuracy of the results. On the contrary, numerical results will show that this choice strongly influence the accuracy of theories that partially satisfy physical constraints, according to [57] to [66].

## 4. Numerical assessments and discussion

Accuracy of previous theories is assessed considering different benchmarks, loading and boundary conditions, some of which exhibit strong layerwise effects. The aim is to evaluate whether accuracy of results is independent on the choice of global and layerwise functions. Conversely, it will be shown that accuracy of theories only partially satisfy physical constraints, is highly dependent on the choices made. Low length-to-thickness ratios with strong layerwise effects in most cases, but also slender ones are analysed, so to assess if findings hold in general. FEA-3D results [72] are used as reference solutions in addition to exact ones where available.

### 4.1 Cases a and b

They concern [90/0] [53] and [0/90/0] [73] simply-supported beams under sinusoidal loading, respectively, so, to assess the correct implementation of theories and to preliminarily prove their accuracy under mild layerwise effects. The results of Tables 7 and 8 demonstrate that the choice of the zig-zag functions is immaterial and can even be omitted, since adaptive theories ZZA, ZZAS4, NOZZG, ZZA\_PP34, ZZA\_PT34, ZZA\_PM34, ZZA\_PMT34, ZZA\_PPM34, ZZA \*, HWZZ, HWZZM, ZZA\_MHR, AT-3D whose coefficients are redefined by full satisfying (3)-(6) are indistinguishable from one another. Considering for example  $u_\alpha$  at  $\zeta = h/2$ , the results of the theories are in sequence

4.5486, 4.5486, 4.4951, 4.5486, 4.5220, 4.5479, 4.5522, 4.5522, 4.5486, 4.5261, 4.5261, 4.5486, 4.5159.

Consequently, they are no longer reported individually in subsequent cases.

It could be noticed that an increase in the number of degrees of freedom as in AT-3D, does not constitute any advantage in terms of accuracy in the case of adaptive theories, hence the maximum accuracy degree has already been obtained through the redefinition of the coefficients. Even theories with a uniform transverse displacement appear rather accurate for these cases. Because of results of  $\sigma_{\zeta\zeta}$  are very accurately predicted by all theories, they are not reported in Tables 7 and 8. In these and following tables, the through thickness representation of a quantity, for which the greatest discrepancies among the theories occur, is reported as an example. Results reported as percentage errors are calculated as  $\left| \frac{q_{ref} - q_{th}}{q_{ref}} \right|$ ;  $q_{ref} = q_{exact/FEA}$ . Accuracy of theories ZZAS1, ZZAS2 and ZZAS3 is strongly dependent by the choice of the representation form and of zig-zag functions as well as of their position across the thickness. Indeed, ZZAS1 (ij=2) is not reported for case b, because it is too inaccurate.

#### 4.2 Cases c to h

Case c and d, retaken respectively from [74] and [44] concern simply-supported laminated beams under sinusoidal loading. Case c is a double core sandwich beam with two thick weak cores and laminated faces, which is simulated as a [(0/90/0)/0/(0/90/0)/0/(0/90/0)] laminate. This case is selected because in [74] it is shown that it cannot be simulated by equivalent single layer theories and so, it is suited for checking if only theories of sections 2.3 to 3.1 or even lower order ones of sections 3.2 to 3.9 could be adequate. Case d is a [0/90/0<sub>5</sub>/90] laminate whose displacements do not satisfy Murakami's rule, so kinematic-based models are not adequate for this case [31].

Cases e and f, retaken from [49] and [31] concern simply supported rectangular sandwich plates under bisinusoidal loading. To enhance layerwise effects, case f has a damaged lower face (components of tensor of elastic moduli  $E_{1111}, E_{1122}, E_{2222}, E_{1212}$  reduced by a factor  $2 \cdot 10^{-1}$ ) and a rather stiffer core. Case g is a propped cantilever sandwich beam under a uniform load retaken from [31], whose peculiarity is to require an accurate piecewise description of the transverse displacement at the supported edge, otherwise stresses are misestimate. Even though a length-to-thickness ratio of 20 is considered, still strong layerwise effects as shown in [31], so, equivalent single layers and other lower-order theories cannot be adequate. Case h [31] is a simply-supported square sandwich plate, that because of a uniform local loading on the upper face, suffers from strong layerwise effects and strongly asymmetric transverse shear stresses across the thickness. Because of lay-up, loading and boundary conditions are symmetric along in-plane directions, the following equalities  $u_\alpha = u_\beta, \sigma_{\alpha\alpha} = \sigma_{\beta\beta}, \sigma_{\alpha\zeta} = \sigma_{\beta\zeta}$  apply.

Regarding case c, adaptive theories are still accurate, while lower-order theories make mistakes from 7 to 55%, except PP23, ZZAS2 (ij=6), ZZAS3 (ij=6), ZZAS1 to ZZAS3 (ij>6), whose errors range from 0.5 to 5.6%. For brevity, these results are not reported.

The results of Tables 9 to 13, that refer to cases d to h, confirm the previous findings, about adaptive and lower order theories. Because of too high percentage errors, ZZAS1 with ij=2 is not reported in Table 10, ZS3\_2 in Tables 10 and 11, ZS1 and PP23 in Table 12. Regarding case e, it should be noticed that most

of lower-order theories, quite accurately predict displacements and stresses in the top face, while quantities in the bottom face are imprecise, because of their simple representation. It could be noticed that ZS1 and ZS1\_1 obtain good predictions of  $\sigma_{\alpha_c}$  (even if less accurate than adaptive theories) for cases d and e, but not for cases f to h and beyond up to m. Although ZS1 contains zig-zag functions and ZS1\_1 is devoid of them, their results are indistinguishable, so, it is demonstrated that the choice of zig-zag functions is immaterial. ZS1\_3 obtains better results than ZS1, ZS2 accurately calculate stresses for cases d, e and g, while ZS3, ZS3\_1 and ZS3\_2 are accurate only for case g. Theory PP23 is not adequate for cases f to h, while ZZAS1, ZZAS2 and ZZAS3 are unreliable for cases e, g and h.

### 4.3 Case i

A simply-supported eleven-layers sandwich beam under sinusoidal loading, is considered, whose laminated faces are made of layers with different thickness and material properties [29], [42]. It represents a very severe test for theories because of the low elastic modulus of the lower face ( $E_{3333}$  reduced by a factor of  $10^{-2}$ ). The results of Figure 1, show that only higher-order adaptive theories can obtain accurate results. Whether the exact solution is not available, 3-D FEA is used as reference solution, here as well as in subsequent cases.

Assumption of uniform transverse displacement is now totally inadequate because anyway stress fields are wrong, even for mixed theories. Redefinition of coefficients is reconfirmed to improve precision, as well as zig-zag layerwise contributions can be explicitly omitted and accuracy of theories PP23, ZS1, ZS1\_1, ZS1\_2, ZS1\_3, ZS1\_4, ZS2, ZS3, ZS3\_1, ZS3\_2, ZZAS1, ZZAS2, ZZAS3 with a partial fulfilment of physical constraints is lost. For the sake of that, they will no longer be reported next, where stronger layerwise effects rise, being too inaccurate. The comparison between MHR and ZZA\_MHR, shows the advantages of redefining amplitudes even when using Murakami's zig-zag function. For case i and subsequent j to m, theories MHR [31] and ZZ [42] are considered, to further support that the redefinition of coefficients dramatically improves accuracy.

### 4.4 Cases j to m

A three-layer simply-supported sandwich plate under bi-sinusoidal loading is considered as case j, where the lower face ( $E_{1111}, E_{1122}, E_{2222}, E_{1212}$  reduced by a factor  $1 \cdot 10^{-2}$ ) and the core are damaged (the core is partially damaged up to  $0.15h$  from below,  $E_{1122}, E_{2222}, E_{1212}, E_{1313}, E_{2323}$  reduced by a factor  $2 \cdot 10^{-1}$ ); as a consequence, strong 3-D effects rise. Case k differs from case i as concerns loading, a step compressive loading on the upper and lower faces of the two halves of the undamaged beam [31] being applied. Nevertheless lay-up is symmetric, displacements and stresses are strongly asymmetric across the thickness, because of loading. Case l is a modified version of beam [75], with simply-supported edges under a two-step loading and a length-to-thickness ratio of 25 is assumed. Additionally, it considers a damaged core ( $E_{1122}, E_{2222}, E_{1212}, E_{1313}, E_{2323}$  reduced by a factor  $1 \cdot 10^{-1}$ ) and upper face ( $E_{1111}, E_{1122}, E_{2222}, E_{1212}$  reduced by a factor  $4 \cdot 10^{-2}$ ). Case m regards a propped-cantilever sandwich beam with a uniform transverse loading on the upper face and a length-to-thickness ratio of 5.714.. An accurate description of transverse deformability is mandatory for this case, otherwise inaccurate results are obtained [31].



The findings of Figures 2 to 5 confirm that the choice functions of representation and the zig-zag layerwise contributions is immaterial for adaptive theories.

In particular, the results of Figure 2 which refer to the case j, show that only in-plane stresses are quite accurately predicted by all theories and the same apply for case k (Figure 3). Results of Figure 4 for case l show that relevant discrepancies among theories still exist also for this rather thin sandwich and that only adaptive theories can be really accurate [31]. Results of case m by Figure 5 highlight the strong layerwise effects of this case, to which follow that displacement and stress fields are strongly asymmetric across the thickness and consequently only higher-order adaptive theories can obtain accurate results.

Note that ZZAM\_P3P4, that impose the same number of equilibrium points of ZZA, but in integral form (computationally more effective), obtains indistinguishable results for cases a to j, while for cases k to m is inaccurate due to numerical problems and for this reason it is not reported.

Computational burden of all theories of this paper (Table 6) is still comparable with that of FSDT, which being inaccurate, as well as the lower-order theories of this paper, do not result in any convenience. This table shows the slightly lower computational costs of lower-order theories, but considering that they are not always accurate, this advantage cannot be exploited.

#### 4.5 Convergence assessments

A convergence study of theories with respect to the expansion order of trial function and of 3-D FEA with respect to meshing is performed in this section. For the former case, results are reported in Table 14 and Figures 6 to 7. Because a quite large number of theories is considered, to limit the amount of data reported just the number of components that ensures convergence for theory ZZA\_PMTP34 are reported since it has the slowest convergence rate among adaptive theories ZZAS4, NOZZG, ZZA\_PP34, ZZA\_PT34, ZZA\_PM34, ZZA\_PPM34, AT-3D. It must be considered that however there is no wide variation of the order of various theories because convergence orders differ only a few units, so the results for ZZA\_PMTP34 are representative of all adaptive theories. They are representative also for all lower-order theories, which however converge to different and less accurate results.

A quite large number of components is required for cases h and m, which refer to a simply-supported plate under localized loading and to a propped cantilever beam, respectively (see Figures 6 to 7), while for cases i to l one component is sufficient (see Table 14). The largest error, which is obtained considering just the first component, is reported in an inset inside Figures 6 to 7, so the convergence curves can be normalized to it, i.e. all curves start from one for reasons of uniformity. To provide more details about the convergence behaviour, stresses and displacements across the thickness are reported for  $M=4$  and 20 (convergence value) in Figure 6, for  $M=5$  and 10 (convergence value) in Figure 7.

In Figure 8  $u_\alpha$  of case b is considered as an example of through the thickness convergence behavior. For this purpose, the converging solution  $u_\alpha$  is firstly expanded in a Fourier series across the thickness, then the number of components necessary to represent the converging solution is determined. Subsequently, the number of components is progressively reduced, then the percentage error committed is calculated. All adaptive theories behave the same way, because of they require the same expansion order for the

convergent solution, and provide the same results for all lower expansion orders considered. Results from  $N = 1$  to  $N = 100$  are reported in Figure 8, but an acceptable error is obtained for  $N = 14$ . Results allow us to guess that if a Fourier series expansion with 14 or more components is used as the representation within theories, whose amplitudes are assumed as d.o.f., similarly to asymptotic and hierarchical theories, the same results of the present adaptive theories with just five d.o.f., would be obtained (it seems to agree with the expansion order declared in [49]).

The error made by the 3-D FEA when the mesh density increases is shown in Figures 9 (as an example case  $m$  is considered). The three numbers indicated in the figure represent the discretization respectively along  $\alpha$ ,  $\beta$ ,  $\zeta$ . Percentage error of stresses and displacements, reported in the figures are evaluated at specific points where less accurate results are obtained, whose position is specified in the figures.

## 5. Concluding Remarks

This study illustrated numerically that the degree of accuracy of higher-order physically-based zig-zag theories (in displacement-based and mixed form) is independent on the choice of global and layerwise functions, once all stress continuity, boundary and equilibrium conditions are enforced at the same time. On the contrary, if coefficients are not redefined or physical constraints are partially satisfied, results are strongly dependent by choices made, as demonstrated by assessments of the lower-order theories, which are sometimes adequate, but not always.

Six new higher-order physically-based adaptive theories have been developed as particularizations of a novel generalized theory which keeps the same advantages of theories previously developed by the authors and enables users to choose arbitrarily the representation and layerwise contributions. In these theories, zig-zag contributions can be omitted (their role can be played by the redefinition of coefficients at layer interfaces) and displacements can be assumed differently each other and for each region across the thickness. Accordingly, these theories become similar to hierarchical and axiomatic/asymptotic theories, but are more efficient, because they require a lower expansion order and only five d.o.f. to get accurate results. Indeed, a piecewise cubic/fourth-order representation for in-plane and transverse displacements, respectively, is sufficient to get the maximal accuracy.

Also an approximate 3-D solution is obtained as particularization of general formulation, whose results are always precise, demonstrating that it can be used as reference solution when exact one is unavailable. Closed form solutions have been presented using Rayleigh-Ritz method and the same trial functions for all theories. Elastostatic benchmarks with distributed/ localized step loading, different boundary conditions and material properties of layers that generate strong layerwise effects were considered. A convergence study has been carried out, in order to determine the minimum expansion order along in-plane directions, beyond which errors don't appreciably decrease.

## Data Availability

All data generated or analyse during the study are included in the manuscript.

## References

- [1] Reddy JN, Robbins DH. Theories and computational models for composite laminates. *Appl Mech Rev* 1994;47(6):147-169.
- [2] Reddy JN. *Mechanics of laminated composite plates and shells: Theory and analysis*, 2nd ed. CRC Press: Boca Raton, United States, 2003.
- [3] Reddy JN, Arciniega RA. Shear deformation plate and shell theories: from Stavsky to present. *Mech Adv Mater Struct* 2004;11:535–582.
- [4] Vasilive VV, Lur'e SA. On refined theories of beams, plates and shells. *J Compos Mat* 1992;26:422–430. <https://doi.org/10.1177/002199839202600405>.
- [5] Lur'e SA, Shumova NP. Kinematic models of refined theories concerning composite beams plates and shells. *Int J Appl Mech* 1996;32:422-430. <https://doi.org/10.1007/BF02313861>.
- [6] Noor AK, Burton SW, Bert CW. Computational model for sandwich panels and shells. *Appl Mech Rev* 1996;49:155–199. <https://doi.org/10.1115/1.3101923>.
- [7] Altenbach H. Theories for laminated and sandwich plates. A review. *Int J Appl Mech* 1998;34:243-252. <https://doi.org/10.1007/BF02256043>.
- [8] Carrera E. Developments, ideas, and evaluations based upon Reissner's mixed variational theorem in the modeling of multilayered plates and shells. *Appl Mech Rev* 2001;54:301-329. <https://doi.org/10.1115/1.1385512>.
- [9] Carrera E. Theories and finite elements for multilayered, anisotropic, composite plates and shells. *Arch Comput Methods Eng* 2002;9(2):87–140. <http://dx.doi.org/10.1007/BF02736649>.
- [10] Carrera E. Historical review of zig-zag theories for multilayered plates and shells. *Appl Mech Rev* 2003;56:1-22. <https://doi.org/10.1115/1.1557614>.
- [11] Carrera E. On the use of the Murakami's zig-zag function in the modeling of layered plates and shells. *Compos. Struct.* 2004;82:541–554, <https://doi.org/10.1016/j.compstruc.2004.02.006>.
- [12] Carrera E, Brischetto S. A survey with numerical assessment of classical and refined theories for the analysis of sandwich plates. *Appl Mech Rev* 2009;62:010803-1–010803-17.

- [13] Qatu MS. Recent research advances in the dynamic behavior of shells: 1989–2000, part 1: laminated composite shells. *Appl Mech Rev* 2002;55(4):325–50. <http://dx.doi.org/10.1115/1.1483079>.
- [14] Qatu MS, Sullivan RW, Wang W. Recent research advances on the dynamic analysis of composite shells: 2000–2009. *Compos Struct* 2010;93(1):14–31. <http://dx.doi.org/10.1016/j.compstruct.2010.05.014>.
- [15] Wanji C, Zhen W. A selective review on recent development of displacement based laminated plate theories. *Recent Patents Mech Eng* 2008;1:29–44.
- [16] Khandan R, Noroozi S, Sewell P, Vinney J. The development of laminated composite plate theories: a review. *J Mater Sci* 2012;47:5901-5910. <https://doi.org/10.1007/s10853-012-6329-y>.
- [17] Kapuria S, Nath JK. On the accuracy of recent global–local theories for bending and vibration of laminated plates. *Compos Struct* 2013;95:163–172. <https://doi.org/10.1016/j.compstruct.2012.06.018>.
- [18] Reddy JN, Liu CF. A higher-order shear deformation theory of laminated elastic shells. *Int J Eng Sci* 1985;23(3):319–30. [http://dx.doi.org/10.1016/0020-7225\(85\)90051-5](http://dx.doi.org/10.1016/0020-7225(85)90051-5).
- [19] Kant T, Kommineni JR. Large amplitude free vibration analysis of cross-ply composite and sandwich laminates with a refined theory and  $C^0$  finite elements. *Comput Struct* 1989;50:123–134.
- [20] Toorani MH, Lakis AA. General equations of anisotropic plates and shells including transverse shear deformations, rotary inertia and initial curvature effects. *J Sound Vib* 2000;237(4):561–615. <http://dx.doi.org/10.1006/jsvi.2000.3073>.
- [21] Mantari JL, Guedes Soares C. Analysis of isotropic and multilayered plates and shells by using a generalized higher-order shear deformation theory. *Compos Struct* 2012;94(8):2640–56. <http://dx.doi.org/10.1016/j.compstruct.2012.03.018>.
- [22] Reddy JN, Barbero EJ, Teply JL. A plate bending element based on a generalized laminate theory. *Int J Numer Meth Eng* 1989;28:2275–2292.
- [23] Rammerstorfer FG, Dorninger K, Starlinger A. Composite and sandwich shells. *Nonlin An Shells Finite Elem* 1992;328:131–194.
- [24] Oh J, Cho M. Higher order zig-zag theory for smart composite shells under mechanical-thermo-electric loading. *Int J Solids Struct* 2007;44(1):100–27. <http://dx.doi.org/10.1016/j.ijsolstr.2006.04.017>.

- [25] Giunta G, Crisafulli D, Belouettar S, Carrera E. Hierarchical theories for the free vibration analysis of functionally graded beams. *Comp Struct* 2011;94:68–74.
- [26] Carrera E, de Miguel AG, Pagani A. Hierarchical theories of structures based on Legendre polynomial expansions with finite element applications. *Int J Mech Sci* 2017;120:286-300.
- [27] Fettahlioglu OA, Steele CR. Asymptotic solutions for orthotropic nonhomogeneous shells of revolution. *J Appl Mech* 1974;41:753–8.
- [28] Mashat DS, Carrera E, Zenkour AM, Al Khateeb SA, Axiomatic/asymptotic evaluation of multilayered plate theories by using single and multi-points error criteria. *Comp Struct* 2013;106:393–406.
- [29] Icardi U, Sola F. Development of an efficient zig-zag model with variable representation of displacements across the thickness. *J Eng Mech* 2014;140:531-541.  
[https://doi.org/10.1061/\(ASCE\)EM.1943-7889.0000673](https://doi.org/10.1061/(ASCE)EM.1943-7889.0000673).
- [30] Icardi U, Urraci A. Free and forced vibration of laminated and sandwich plates by zig-zag theories differently accounting for transverse shear and normal deformability. *Aerosp MDPI* 2018;5:108.  
<https://doi.org/10.3390/aerospace5040108>.
- [31] Icardi U, Urraci A. Novel HW mixed zig-zag theory accounting for transverse normal deformability and lower-order counterparts assessed by old and new elastostatic benchmarks. *Aer Sci Tech* 2018;80:541-571. <https://doi.org/10.1016/j.ast.2018.07.040>.
- [32] Kapuria S, Dumir PC, Jain NK. Assessment of zig-zag theory for static loading, buckling, free and forced response of composite and sandwich beams. *Compos Struct* 2004;64:317–327.  
<https://doi.org/10.1016/j.compstruct.2003.08.013>.
- [33] Zhen W, Wanji C. Free vibration of laminated composite and sandwich plates using global–local higher-order theory. *J Sound Vib* 2006;298:333–349. <https://doi.org/10.1016/j.jsv.2006.05.022>.
- [34] Burlayenko VN, Altenbach H, Sadowski T. An evaluation of displacement-based finite element models used for free vibration analysis of homogeneous and composite plates. *J Sound Vib* 2015;358:152–175. <https://doi.org/10.1016/j.jsv.2015.08.010>.
- [35] Jun L, Xiang H, Li Xiaobin L. Free vibration analyses of axially loaded laminated composite beams using a unified higher-order shear deformation theory and dynamic stiffness method. *Compos Struct* 2016;158:308–322. <https://doi.org/10.1016/j.compstruct.2016.09.012>.

- [36] Tessler A, Di Sciuva M, Gherlone M. A refined zigzag beam theory for composite and sandwich beams. *J Compos Mat* 2009;43:1051–1081. <https://doi.org/10.1177/0021998308097730>.
- [37] Di Sciuva M. A refinement of the transverse shear deformation theory for multilayered orthotropic plates. *L’Aerot Miss Spaz* 1984;62:84–92.
- [38] Cho M, Parmerter R. Efficient higher order composite plate theory for general lamination configurations. *AIAA J* 1993;31:1299-1306. <https://doi.org/10.2514/3.11767>.
- [39] Nguyen SN, Lee J, Cho M. Efficient higher-order zig-zag theory for viscoelastic laminated composite plates. *Int J Sol Str* 2015;62:174-185. <https://doi.org/10.1016/j.ijsolstr.2015.02.027>.
- [40] Murakami H, Maewal A, Hegemier GA. A Mixture Theory With a Director for Linear Elastodynamics of Periodically Laminated Media. *Int J Sol Str* 1981;17:155–173. [https://doi.org/10.1016/0020-7683\(81\)90072-X](https://doi.org/10.1016/0020-7683(81)90072-X).
- [41] Murakami H. Laminated composite plate theory with improved in-plane responses. *ASME Appl Mech* 1986;53:661–666. 10.1115/1.3171828.
- [42] Icardi U. Higher-order zig-zag model for analysis of thick composite beams with inclusion of transverse normal stress and sublaminates approximations. *Compos Part B* 2001;32:343-354. [https://doi.org/10.1016/S1359-8368\(01\)00016-6](https://doi.org/10.1016/S1359-8368(01)00016-6).
- [43] Gherlone M. On the use of zigzag functions in equivalent single layer theories for laminated composite and sandwich beams: a comparative study and some observations on external weak layers. *ASME Appl Mech* 2013;80:1-19. 10.1115/1.4023690.
- [44] Groh RMJ, Weaver PM. On displacement-based and mixed-variational equivalent single layer theories for modeling highly heterogeneous laminated beams. *Int J Solids Struct* 2015;59:147-170. <https://doi.org/10.1016/j.ijsolstr.2015.01.020>.
- [45] Kim JS, Cho M. Enhanced first-order theory based on mixed formulation and transverse normal effect. *Int J Sol Struct* 2007;44:1256–1276.
- [46] Barut A, Madenci E, Tessler A. A refined zigzag theory for laminated composite and sandwich plates incorporating thickness stretch deformation, in: *Proc. 53rd AIAA/ASME/ASCE/AHS/ACS Structures, Structural Dynamics and Materials Conference, Hawaii, 2012*.

- [47] Iurlaro L, Gherlone M, Di Sciuva M. The (3, 2)-mixed refined zigzag theory for generally laminated beams: theoretical development and  $C^0$  finite element formulation. *Int J Sol Struct* 2015;19:73–74.
- [48] Zhen W, Wanji C. A global higher-order zig-zag model in terms of the HW variational theorem for multi-layered composite beams. *Compos Struct* 2016;158:128–136.
- [49] Brischetto S, Carrera E, Demasi L. Improved response of asymmetrically laminated sandwich plates by using Zig-Zag functions. *J Sand Struct Mat* 2009;11:257-267.  
<https://doi.org/10.1177/1099636208099379>.
- [50] Li XY, Liu D. Generalized laminate theories based on double superposition hypothesis. *Int J Num Meth Eng*. 1997;40:197–212. [https://doi.org/10.1002/\(SICI\)1097-0207\(19970415\)40:7<1197::AID-NME109>3.0.CO;2-B](https://doi.org/10.1002/(SICI)1097-0207(19970415)40:7<1197::AID-NME109>3.0.CO;2-B).
- [51] Zhen W, Wanji C. A study of global–local higher-order theories for laminated composite plates. *Compos Struct* 2007;79:44–54. <https://doi.org/10.1016/j.compstruct.2005.11.027>.
- [52] Shariyat M. A generalized global-local higher order theory for bending and vibration analyses of sandwich plates subjected to thermo-mecanical loads. *Int J MechSci* 2010;52:495-514.
- [53] Catapano A, Giunta G, Belouettar S, Carrera E. Static analysis of laminated beams via a unified formulation. *Compos Struct* 2011;94:75-83. <https://doi.org/10.1016/j.compstruct.2011.07.015>.
- [54] de Miguel AG, Carrera E, Pagani A, Zappino E. Accurate Evaluation of Interlaminar Stresses in Composite Laminates via Mixed One-Dimensional Formulation. *AIAA J* 2018;56:4582-4594.  
<https://doi.org/10.2514/1.J057189>.
- [55] Icardi U, Sola F.  $C^0$  fixed degrees of freedom zigzag model with variable in-plane and out-of plane kinematics and quadrilateral plate element. *J Aero Eng* 2014;28(5).  
[https://doi.org/10.1061/\(ASCE\)AS.1943-5525.0000472](https://doi.org/10.1061/(ASCE)AS.1943-5525.0000472).
- [56] Vescovini R, Dozio LA. Variable-Kinematic Model for Variable Stiffness Plates: Vibration and Buckling Analysis. *Compos Struct* 2016;142:15-26. doi:10.1016/j.compstruct.2016.01.068.
- [57] Surana K, Nguyen S. Two-dimensional curved beam element with higher order hierarchical transverse approximation for laminated composites. *Compos Struct* 1990;36:499-511.  
[https://doi.org/10.1016/0045-7949\(90\)90284-9](https://doi.org/10.1016/0045-7949(90)90284-9).

- [58] Shimpi RP, Ghugal YM. A layer-wise trigonometric shear deformation theory for two-layered cross-ply laminated beams. *J Reinf Plast Compos* 1999;18:1516–42.
- [59] Rao MK, Desai YM, Chitnis MR. Free vibrations of laminated beams using mixed theory. *Compos Struct* 2001;52:149-160. [https://doi.org/10.1016/S0263-8223\(00\)00162-8](https://doi.org/10.1016/S0263-8223(00)00162-8).
- [60] Karama M, Afaq K, Mistou S. Mechanical behaviour of laminated composite beam by the new multi-layered laminated composite structures model with transverse shear stress continuity. *Int J Solids Struct* 2003;40:1525-1546. [https://doi.org/10.1016/S0020-7683\(02\)00647-9](https://doi.org/10.1016/S0020-7683(02)00647-9).
- [61] Vidal P, Polit O. Assessment of the refined sinus model for the non-linear analysis of composite beams. *Compos Struct* 2009;87:370-381. <https://doi.org/10.1016/j.compstruct.2008.02.007>.
- [62] Jun L, Hongxing H. Dynamic stiffness analysis of laminated composite beams using trigonometric shear deformation theory. *Compos Struct* 2009;89:433-442. <https://doi.org/10.1016/j.compstruct.2008.09.002>.
- [63] Rodrigues JD, Roque CMC, Ferreira AJM, Carrera E, Cinefra M. Radial basis functions-finite differences collocation and a unified formulation for bending, vibration and buckling analysis of laminated plates, according to Murakami's zig-zag theory. *Compos Struct* 2011;93:1613–1620. <https://doi.org/10.1016/j.compstruct.2011.01.009>.
- [64] Mantari J, Oktem A, Soares CG. A new higher order shear deformation theory for sandwich and composite laminated plates. *Compos Part B* 2012;43:1489-1499. <https://doi.org/10.1016/j.compositesb.2011.07.017>.
- [65] Rodrigues JD, Roque CMC, Ferreira AJM, Cinefra M, Carrera E. Radial basis functions-differential quadrature collocation and a unified formulation for bending, vibration and buckling analysis of laminated plates, according to Murakami's Zig-Zag theory. *Comput Struct* 2012;90–91:107–115.
- [66] Carrera E, Filippi M, Zappino E. Laminated beam analysis by polynomial, trigonometric, exponential and zig-zag theories. *Euro J Mech A/Solids* 2013;41:58-69.
- [67] Icardi U, Urraci A. Free Vibration of flexible soft-core sandwiches according to layerwise theories differently accounting for the transverse normal deformability. *Lat A J Sol Struct* In Press.
- [68] Icardi U, Urraci A. Elastostatic assessment of several mixed/displacement-based laminated plate theories, differently accounting for transverse normal deformability. *Aerosp Sc Tech* In Press.



[69] Urraci A, Icardi U. Zig-zag theories differently accounting for layerwise effects of multilayered composites. *Int J Eng Res Sci* 2019;5(6):21-42. 10.5281/zenodo.3264035.

[70] Urraci A, Icardi U. New 3-D zig zag theories: elastostatic assessment of strategies differently accounting for layerwise effects of laminated and sandwich composites. *Int J Eng Res Appl* 2019;9(7):1-25. 10.9790/9622-0907050125.

[71] Urraci A, Icardi U. Approximate 3-D model for analysis of laminated plates with arbitrary lay-ups, loading and boundary conditions. *Int J Eng Res Sci* 2019;5(7):21-39. 10.5281/zenodo.3356468.

[72] Icardi U, Atzori A. Simple, efficient mixed solid element for accurate analysis of local effects in laminated and sandwich composites. *Adv Eng Soft* 2004;35:843-859.  
<https://doi.org/10.1016/j.advengsoft.2004.07.003>.

[73] Pagano NJ. Exact solutions for composite laminates in cylindrical bending. *J Compos Mater* 1969;3:398-411. <https://doi.org/10.1177/002199836900300304>.

[74] Zhen W, Rui M, Wanji C. A C0 three-node triangular element based on preprocessing approach for thick sandwich plates. *J Sand Struct Mat* 2017;0:1-23. <https://doi.org/10.1177/1099636217729731>.

[75] Mattei O, Bardella L. A structural model for plane sandwich beams including transverse core deformability and arbitrary boundary conditions. *Euro J Mech Part A Solids* 2016;58:172-186.  
<https://doi.org/10.1016/j.euromechsol.2016.01.015>.