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Effects of cell quality in Grid Boundary Layer on the simulated flow around a square cylinder

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Abstract

The flow around a square cylinder is widely studied as a paradigmatic case in bluff body aerodynamics. The effects of several physical parameters of the setup, and the errors induced by turbulence models, numerical schemes and grid density have been emphasized in a huge number of studies during the past two decades. Surprisingly, the effects of the grid quality on such a class of flow has been overlooked. The lack of a shared approach and suggested best practices for high-quality grid generation among scholars and practitioners follows. The present study aims at filling this gap. The cell skewness and non-orthogonality are adopted as metrics of the grid quality. The errors induced by poor quality cells and the possible corrective measures are discussed in a Finite Volume Method framework. The effects of the cell quality on the simulated flow are systematically evaluated by a parametrical study including four different types of grid boundary layer. The obtained results are compared among them and discussed in terms of instantaneous and time averaged flow fields, stress distribution at wall, and aerodynamic coefficients. Both the overall modelling error and the skewness-induced one are evaluated with reference to a huge number of data collected from previous studies. The local error induced by few, moderately skewed, near-wall cells upwind the cylinder propagates windward because

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of the convection-dominated problem, and globally affects the boundary layer separation and the vortex shedding in the wake. Skewness around the trailing edge only affects the flow to a lower extent. The skewness error on bulk aerodynamic coefficients may largely prevails on the overall modelling error, in spite of the very simple turbulence model deliberately adopted in the study. Hybrid grid boundary layer made of structured cells along the cylinder sides and unstructured ones around its edges provides results analogous to the ones obtained with a fully orthogonal grid, in spite of some clusters of few skewed cells far from the wall. Hybrid grid boundary layer is recommended as a fine balance between accuracy and flexibility in grid generation, when full orthogonal grid boundary layer is not feasible around real-world engineering applications having complicate geometries with multiple obtuse or acute edges.

Keywords: Finite Volume Method, grid boundary layer, grid quality, skewness, square cylinder

1 1. Introduction

The high-Reynolds number, turbulent flow around rectangular cylinders is 2 considered as a paradigmatic one around bluff bodies, of interest both for fundamental research and applications. In spite of the simple and nominally twodimensional geometry, the flow is intricate, characterized by boundary layer separation, its possible reattachment depending on the ratio of the alongwind 6 to crosswind dimensions, Kelvin-Helmholtz instability of the detached shear layer, Von-Karman-like vortex shedding along the wake. Such underlying flow regimes provide useful information on the aerodynamics of a wide range of bluff bodies of interest in civil engineering (e.g. long-span bridge decks or high-rise 10 buildings) as well as in other engineering areas. The importance attached to 11 such flows by the scientific community is testified by some international bench-12 marks, such as the ones on the flow around 1:1 square cylinder (Rodi, 2004; 13 Rodi et al., 1997), and around 5:1 rectangular cylinder (Bruno and Salvetti, 14 2017; Bruno et al., 2014). 15

The studies within the benchmarks above have testified the high sensitivity 16 of the flow to both physical parameters of the setup, and to errors induced by 17 the components of the computational model. Among the former, special at-18 tention was paid for instance to the cylinder edge roundness (Tamura et al., 19 1998; Riberio, 2011; Cao and Tamura, 2017; Rocchio et al., 2020), Reynolds 20 number (Scruton et al., 1971; Mannini et al., 2010; Schewe, 2013), and to other 21 freestream conditions (e.g. Vickery, 1966; Lee, 1975; Mariotti et al., 2016; Cao 22 and Tamura, 2018). Among the latter, the effects of turbulence models on the 23 simulated flow have been emphasized in a huge number of studies (e.g. Franke 24 and Rodi, 1991; Kato and Launder, 1993; Rodi, 1997; Lee, 1997; Bosch and 25 Rodi, 1998; Sohankar et al., 2000; Ke, 2019). In their wake, the conscious and 26 adapted application of different turbulence models is currently widespread in 27 engineering practice. Even if to a minor extent, also the effects of the numerical 28 schemes on the simulated flow were scrutinized by a number of studies, with 29 special emphasis on the discretization of the convective terms (Shyv et al., 1992; 30 Lee, 1997; Tamura et al., 1998; Cao and Tamura, 2016; Zhu et al., 2020). 31

More recently, the exponential growth of HPC facilities is allowing the sys-32 tematic evaluation of the effects of the grid density, both in the vertical along-33 wind x - y plane (Cao and Tamura, 2016) and along the spanwise z-direction 34 (Bruno et al., 2012; Cao and Tamura, 2016; Cao et al., 2020; Zhang and Xu, 35 2020). The attention of other authors focused on Adaptive Mesh Refinement 36 in Finite Element Method (see e.g. Hoffman, 2005; Berrone and Marro, 2009) 37 and Finite Volume Method (FVM) (see e.g. Antepara et al., 2015). The errors 38 induced by poor grid quality in FVM are well known in fundamental research on 39 computational methods since the pioneering PhD Theses of Muzaferija (1994) 40 and Jasak (1996), up to the recent studies by e.g. Ahipo and Traoré (2009); 41 Traoré et al. (2009); Juretić and Gosman (2010); de Oliveira Samel Moraes 42 et al. (2013); Denner and van Wachem (2014, 2015). Conversely, grid quality 43 effects on the simulated flow around bluff bodies are surprisingly overlooked in 44 scientific and technical literature. In particular, the current and very pervasive 45 practice in the simulation of flow around cylinders implies that fully orthogonal 46

structured grids are generated by extrusion with constant grid spacing in the 47 spanwise direction, but non orthogonal and skewed grids are inevitably gener-48 ated in the vertical alongwind plane orthogonal to the cylinder axis. To our best 49 knowledge, only a few number of studies partially cover such an issue. Nakane 50 (2013) applied FVM with orthogonal and non-orthogonal grids to the low-Re 51 (Re=550, 3000) transient flow around a circular cylinder, with focus on the early 52 stage of the symmetrical steady reversed flow in the near wake. Haque et al. 53 (2016) paid attention to the effects of the cell height growing ratio across the 54 Grid Boundary Layer (GBL) on the high-Re flow around rectangular cylinders 55 and hexagonal bridge decks. The grid quality across the GBL is expected to 56 be of paramount importance, because of the high gradient of the velocity and 57 turbulent variables in the boundary layer. Shortage in studies on the effects 58 of grid quality in GBL results in the lack of a shared approach and suggested 59 best practices among scholars and practitioners. For instance, Figure 1 provides 60 some examples of different GBLs adopted in recent computational studies on 61 the aerodynamics of 1:1 and 5:1 rectangular cylinders. Even if a structured

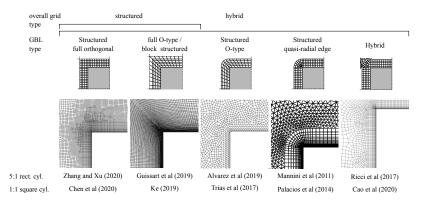


Figure 1: Types of GBL adopted in recent studies within benchmarks on the aerodynamics of 1:1 square and 5:1 rectangular cylinders

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orthogonal GBL with quadrangular cells is generated in all studies along the
 cylinder sides, they significantly differ in the meshing strategy around the sharp

(or slightly rounded) corners, in the shape of the cells around them, and in the 65 grid quality in turn. A non necessary exhaustive GBL categorization is ten-66 tatively proposed in the figure, in the absence of a common nomenclature in 67 literature. All meshing strategies at corners are structured, except for the 'Hy-68 brid GBL', where orthogonal boundary-fitted grid along the straight sides of the 69 cylinder alternates with unstructured grid in the quadrants departing from the 70 sharp corners. The latter inevitably includes almost regular, quasi orthogonal 71 and skewness-free cells in the first layers around the cylinder corners because 72 of the strong two-side constraint of the discretization of the adjacent edges of 73 the structured parts. Low quality cells take place moving away from the cylin-74 der corners along and around the bisector of the quadrant. A single, largely 75 prevailing GBL meshing approach among the above categories does not exist in 76 research and engineering practice. For instance, Figure 2 provides statistics on 77 GBLs adopted in 23 studies published in the last decade on the aerodynamics 78 of the 5:1 rectangular cylinder.

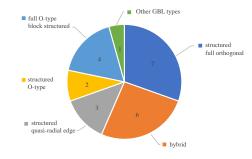


Figure 2: Statistics on types of GBL adopted in literature for the Benchmark on the Aerodynamics of a Rectangular 5:1 Cylinder (BARC)

79

An even larger variability of GBL meshing strategies occurs for other cylinderlike, real-world engineering applications having more complicate geometries with multiple obtuse or acute edges, such as bridge deck aerodynamics. For instance, Tang et al. (2019) adopt a Hybrid GBL, Nieto et al. (2015) a block-structured grid, Mannini et al. (2016) a quasi radial GBL, being full orthogonal GBL unfeasible. The grid-induced errors are also expected to vary correspondingly. The effects of low-quality grids could be even larger in fluid-structure interaction applications, such as bridge or cable aeroelasticity, where highly distorted cells can arise from large amplitude pitching motion of the cylinder (see e.g. Lee et al., 2016, Fig.2).

This study aims to shed some light on the errors and effects of different 90 GBLs on the simulated flow around 1:1 square cylinder, and to compare such 91 grid-induced errors with the modelling errors. Besides this introduction, the 92 paper is organized into four more sections. In Section 2 the overall adopted 93 computational model is shortly recalled, while spatial grid quality metrics and 94 effects are detailed. The setup of the application is described in Section 3. 95 The obtained results are commented in Section 4. Finally, in Section 5 final 96 conclusions are drawn, and some research perspectives are briefly sketched. 97

98 2. Computational model

In the following, the adopted computational model is described in terms of qq turbulence model, boundary conditions, numerical approach, and spatial grid 100 quality. Basic, well established and widely used turbulence model and numerical 101 approach are deliberately adopted in a 2D domain. They are expected to cor-102 rectly reproduce the flow of interest in qualitative terms (Lee, 1997), although 103 with quantitative errors. Departures from top quality grid and induced errors 104 are detailed, the study being intended to quantitatively compare the overall er-105 ror model with the one induced by low-quality GBLs in the vertical alongwind 106 plane orthogonal to the cylinder axis. 107

108 2.1. Turbulence model

The incompressible, turbulent, separated, unsteady flow around the 2D section is modeled by the classical Unsteady Reynolds Averaged Navier-Stokes (URANS) equations, which in indicial form read:

$$U_{i,i} = 0, (1)$$

112

$$U_{i,t} + U_j U_{i,j} = -\frac{1}{\rho} P_{,i} + \left(\nu \left(U_{i,j} + U_{j,i}\right) - R_{ij}\right)_{,j}, \qquad (2)$$

where U_i is the averaged velocity component in the *i*-th direction, P is the 113 averaged pressure, ρ the air density, and ν its kinematic viscosity, $R_{ij} = \overline{u_i u_j}$ 114 the Reynolds stress, and u the velocity turbulent fluctuations. The $k - \varepsilon$ RNG 115 (Yakhot et al., 1992) turbulence model is used to close the URANS equations. 116 The Reynolds stress is expressed by the well-know Boussinesq assumption as 117 $R_{ij} = -2\nu_t S_{ij}$, where S_{ij} is the mean field strain rate, and the turbulent vis-118 cosity ν_t is expressed as $\nu_t = C_\mu \frac{k^2}{\varepsilon}$ in terms of the turbulent kinetic energy k 119 and its dissipation rate ε . Their transport equations are 120

$$k_{,t} + U_j k_{,j} = -R_{ij} U_{i,j} + \left(\frac{\nu_t}{\sigma_k} k_{,i}\right)_{,i} - \varepsilon, \qquad (3)$$

121

$$\varepsilon_{,t} + U_j \varepsilon_{,j} = -C_{\varepsilon_1} \frac{\varepsilon}{k} R_{ij} U_{i,j} + \left(\frac{\nu_t}{\sigma_{\varepsilon}} \varepsilon_{,i}\right)_{,i} - C_{\varepsilon_2}^* \frac{\varepsilon^2}{k},\tag{4}$$

where $C_{\varepsilon_2}^* = C_{\varepsilon_2} + C_{\mu} \eta^3 \frac{(1-\eta/\eta_0)}{1+\beta\eta^3}$, and $\eta = \frac{k}{\varepsilon} \sqrt{2S_{ij}S_{ij}}$ is the ratio between the 122 turbulence time scale and the time scale of the mean field strain. The model 123 constants are $C_{\mu} = 0.085$, $C_{\varepsilon_1} = 1.42$, $C_{\varepsilon_2} = 1.68$, $\eta_0 = 4.38$, $\beta = 0.012$, 124 $\sigma_k = \sigma_{\varepsilon} = 1.393$. The cell-centered nodal values of the flow variables adjacent 125 to the wall are obtained by the standard wall function approach (Launder and 126 Spalding, 1974). Dirichlet boundary condition (b.c.) on the velocity field and 127 on the turbulent variables are imposed at the inlet. Neumann b.c. on the normal 128 component of the stress tensor, as well as the same Dirichlet b.c. on k and ε , 129 are imposed at the outlet. Periodic b.c. are imposed on both the upper-lower 130 boundaries. No-slip b.c. are imposed at the cylinder wall. 131

132 2.2. Numerical approach

A classic overall 2nd order approach in the framework of the Finite Volume 133 Method is adopted to discretize the governing equations above. The cell-centre 134 values of the variables are interpolated at face locations using the classic second-135 order Central Difference Scheme (CDS) for the diffusive terms. The convection 136 terms are discretized by means of the QUICK scheme (Leonard, 1979). The 137 fully implicit second-order Euler scheme is adopted for time discretization. The 138 SIMPLE algorithm (Issa, 1986) is used for pressure-velocity coupling. The code 139 Ansys Fluent^(C) v.18 is used. 140

141 2.3. Grid quality effects

¹⁴² Non-orthogonality and skewness are common issues on non-Cartesian, ar ¹⁴³ bitrary grids. A structured radial edge and an O-type GBLs are considered in Figure 3 as examples. Both non-orthogonality and skewness are qualities

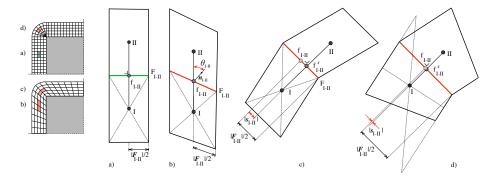


Figure 3: Face orthogonality $\theta = 0$ (a), non-orthogonality $\theta \neq 0$ (b), and skewness s (c,d) of cells around a sharp edge in different types of GBL

144

that refer to each face F_{I-II} shared between adjacent cells with centers I and II. 145 Some pairs of cells are highlighted in Figure 3 for the sake of clarity. Non-146 orthogonality refers to the angle $\theta_{\text{I-II}}$ between the normal vector $n_{\text{I-II}}$ and the 147 vector connecting the two cell centers. The face is orthogonal if $\theta_{I-II} = 0$ (Fig. 148 3-a, -c, -d), non-orthogonal otherwise (Fig. 3-b). Skewness refers to the vector 149 $s_{\text{I-II}}$ from the face mid-point $f_{\text{I-II}}$ to the interpolation point $f_{\text{I-II}}^s$, the latter being 150 defined as the intersection between the face and the vector connecting the two 151 cell centers. The face is not skewed if $|\mathbf{s}_{\text{I-II}}| = 0$ (Fig. 3-a,b), skewed otherwise 152 (Fig. 3-c,d). The examples in Figure 3 clearly underline that rectangular cells 153 are perfectly orthogonal and not skewed, and that non-orthogonality does not 154 necessarily implies skewness, and vice-versa. The above metrics can be made 155 scalar and dimensionless as $\theta^*_{\text{I-II}} = 2\theta_{\text{I-II}}/\pi$ and $s^*_{\text{I-II}} = 2|s_{\text{I-II}}|/|F_{\text{I-II}}|$, where 156 $|F_{I-II}|$ is the length of the face F_{I-II} (Fig. 3). Both dimensionless metrics take 157 values larger than unit for degenerate cells and interpolation points outside the 158 face, respectively. 159



Both non-orthogonality and skewness are *potential sources of errors* in the

¹⁶¹ numerical fluxes evaluated by FVM. *Non-orthogonality-induced error* on the dif-¹⁶² fusive terms can be pointed out by splitting the face surface vector $F_{\text{I-II}}$ in the ¹⁶³ component $\Delta \parallel \mathbf{d}_{\text{I-II}}$ and in a non-orthogonal one **k** (Fig. 4-a, after Jasak, ¹⁶⁴ 1996). Whatever the chosen direction of **k** is, the error induced on the diffusive ¹⁶⁵ flux of a generic variable ϕ reads in vectorial form

$$E_{no}^d = \nabla \cdot (\nu_{no} \nabla \phi) \tag{5}$$

where $\nu_{no} = U_{f_{I-II}} \cdot \mathbf{k}$ is the so-called non-orthogonal viscosity, and $U_{f_{I-II}}$ is the velocity vector at the face mid-point.

Skewness-induced error can affect the convective terms, and results in a spurious
 non-physical diffusion flux evaluated by Jasak (1996) as

$$E_{sk}^c = \nabla \cdot (\nu_{sk} \nabla \phi) \tag{6}$$

where $\nu_{sk} = U_{f_{I-II}} \cdot s$ is the so-called skewness viscosity. In other terms, grid skewness reduces the accuracy of interpolation to first order, and introduces a diffusion-like error in the discretized convective flux, analogously to the wellknown leading truncation error of the first-order upwind scheme.

In short, both non-orthogonality and skewness indroduce non-physical diffusion, 174 if not properly dealt. In an engineering perspective such diffusive effects are not 175 negligible for convection-dominated, high Reynolds number flow, where physical 176 diffusive effects play a minor role. In other terms, both grid-induced viscosities 177 can cumulate the kinematic viscosity and the modeled turbulent one, resulting 178 in a simulated Re number lower than the one that characterizes the actual flow. 179 Two approaches can be adopted to counter this problem. *High-quality grid* 180 generation is intended to eliminate the error source, i.e. to reduce the amplitude 181 of the geometrical metrics θ and s, and the induced errors in turn. High quality 182 grid may be generated a priori, or obtained a posteriori by grid adaptation. 183 Whatever the adopted technique is, such an approach is the most decisive, even 184 if generation/adaptation criteria are required. Corrective terms in interpolation 185 schemes are intended to mitigate the effects of low-quality grids, i.e. to reduce 186 the errors (eq.s 5 and 6) without eradicate their causes. 187

¹⁸⁸ Non-orthogonal correction strategies move from the surface vector splitting $F_{I-II} = \Delta + \mathbf{k}$ initially proposed by Jasak (1996) (Fig. 4-a).

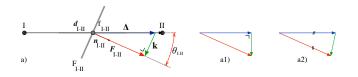


Figure 4: Approache for correction of non-orthogonality (after Jasak, 1996)

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¹⁹⁰ The diffusive flux for the generic variable ϕ can be approximated as

$$\boldsymbol{F}_{\text{I-II}} \cdot (\nabla \phi)_{\text{f}_{\text{I-II}}} = \boldsymbol{\Delta} \cdot (\nabla \phi)_{\text{f}_{\text{I-II}}} + \mathbf{k} \cdot (\nabla \phi)_{\text{f}_{\text{I-II}}} \approx |\boldsymbol{\Delta}| \frac{\phi_{\text{II}} - \phi_{\text{I}}}{|\mathbf{d}_{\text{I-II}}|} + \mathbf{k} \cdot (\widetilde{\nabla \phi})_{\text{f}_{\text{I-II}}}, \quad (7)$$

where the classic CDS is used for the orthogonal term. The non-orthogonal 191 correction second term depends on both the definition of \mathbf{k} , and the numer-192 ical approximation of $(\nabla \phi)_{f_{I-II}}$. Different explicit non-orthogonal correction 193 schemes have been proposed in literature, e.g. Jasak (1996); Ahipo and Traoré 194 (2009); Traoré et al. (2009). Three possible choices for **k** are depicted in Fig-195 ures 4(a), (a1), (a2), where the former minimizes the correction. The iterative 196 numerical approximation $(\widetilde{\nabla \phi})_{\mathbf{f}_{1,\mathbf{H}}}$ is generally inspired to the precursor ideas 197 in Khosla and Rubin (1974) and Muzaferija (1994). Generally speaking, the 198 non-orthogonality correction let to preserve the spatial second order accuracy. 199 However, when its contribution is larger than the orthogonal one for highly 200 non-orthogonal faces, the boundedness of the solution is no longer guaranteed 201 (Jasak, 1996). In order to ensure boundedness, the non-orthogonal correction 202 must be bounded or even discarded and the scheme becomes no longer second 203 order accurate (de Oliveira Samel Moraes et al., 2013). 204

Also skewness correction strategy is intended to correct standard interpolation schemes for the quadrature values by additional term(s) accounting for s. Starting from the pioneering work of (Jasak, 1996) where the skewness problem has been rigorously presented, some proposals for skewness correction have been recently proposed in literature, e.g. de Oliveira Samel Moraes et al. (2013); ²¹⁰ Denner and van Wachem (2014, 2015).

While non-orthogonal correction is widely implemented in industrial codes such as OpenFoam or Fluent, the same can not be said about the skewness correction (de Oliveira Samel Moraes et al., 2013). In the light of this, the present study aims at evaluating different strategies of GBL generation to reduce the skewness geometrical metric |s|, while GBL non-orthogonality is fixed according to the correction given in eq. (7).

217 3. Setup of the study

The adopted flow setup closely follows the one prescribed by the Ercoftac 218 benchmark Ufr 2-02, detailed by Rodi (2004) in the QNET-CFD Knowledge 219 Base Wiki (Rodi, 2012). The benchmark adopts as reference experimental tests 220 the ones detailed in Lyn and Rodi (1994) and Lyn et al. (1995). The incoming 221 flow is characterized by Reynolds number $Re = UD/\nu = 2.2e + 4$, where U is 222 the free stream velocity and D the cylinder chord, incidence $\alpha = 0$, turbulence 223 intensity It = 2%, turbulent length scale Lt = 0.5D. The cylinder cross section 224 has sharp edges and smooth surfaces. The dimensionless time step is set $\Delta t^* =$ 225 0.02tU/D. The simulation is extended over $T \approx 300$ dimensionless time units 226 in order to have a long enough statistical sample to secure converged statistics, 227 after having excluded the initial transient. The dimensions of the 2D domain 228 are given in Figure 5(a). The origin of the Cartesian coordinate is located at 229 the center of the cylinder upwind face. The spatial grid is overall hybrid, being 230 composed by both quadrangular and triangular cells, and by structured and 231 unstructured partitions. A close-up view of the grid around the cylinder and its 232 close wake is shown in Figure 5(b). In addition to usual strategies, subdomain 233 partitioning and corresponding grid blocks are intended to ensure a dense, top-234 quality grid made of square cells ($\Delta_x = \Delta_y = 0.042D$) in the wake region, and 235 to keep the grid outside the GBL unchanged for all cases. 236

Four types of GBLs are generated in a D-tick square annular partition around the cylinder, as detailed in Figure 5(c). All GBLs share some common

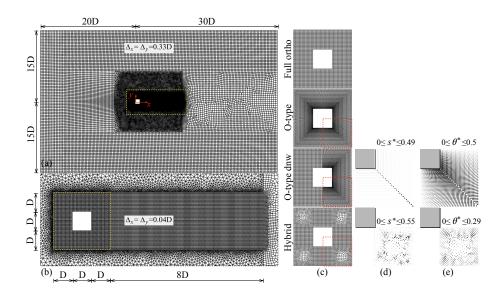


Figure 5: Computational domain (a), GBL and wake subdomain (b), GBL types (c) and their dimensionless skewness (d) and non-orthogonality (e)

features, in order to allow a proper comparative parametric study: cylinder side 239 uniformly discretized; height of the first cell at wall equal to $\Delta n/D = 1/24$; 240 growing ratio equal to unit (nil grid gradient) and number of cells equal to 24 241 across the structured parts of the GBL. Δn is set to comply the requirements 242 of wall-function-based near-wall treatment about the height of the control vol-243 umes at wall: the wall unit $n^+ = nu_*/\nu$ results in the range $25 \le n^+ \le 55$. The 244 generated GBLs share the discretization of the quadrant edges, but differ in the 245 type of grid. The 'Full ortho' GBL has the same top quality grid a in the wake. 246 The 'O-type' one is characterized by a block-structured non-orthogonal grid. 247 The 'O-type dnw' GBL is a mix of the previous ones, where O-type grid is lim-248 ited to the downwind half of the annular partition. In contrast to the previous 249 GBLs, a 'Hybrid' GBL is not uniquely defined by the discretization of the outer 250 edges of the quadrants, but it necessarily depends also on the specific algorithm 251 adopted for grid generation. In the present study, a paving, advancing-front-252 based algorithm is employed (Blacker and Stephenson, 1991). The resulting, 253

specific realization of the Hybrid GBL shows the general distinctive features of 254 this class of GBLs as detailed in the introduction. However, in this specific case 255 the density and quality of the cells do not monotonically decrease moving away 256 from the wall as usually (see e.g. Fig. 1), because of nil grid gradient across 257 the structured parts of the GBL. Skewness and non-orthogonality are nil every-258 where in the GBL for the 'Full ortho' GBL and the upwind part of the 'O-type 259 dnw' one. The spatial patterns of dimensionless skewness and non-orthogonal 260 metrics are given by scatterplots in Figures 5(d) and (e), respectively, for the 261 other cases and limited to the quadrant enclosed by the red dashed line. The 262 s^* pattern in O-type GBLs groups skewness $s^* \neq 0$ along the diagonal only, 263 while $\theta^* \neq 0$ is distributed in the rest of the field. Significant values of both 264 s^* and θ^* are more scattered in the generated Hybrid GBL, with the highest 265 values in correspondence of the transition between dense and coarse grid. In 266 particular, the present Hybrid GBL shows nearly nil skewness in the first 3 cell 267 rows around the cylinder corners (corresponding to a width of 1/8D), because 268 of the constraints imposed by the adjacent edges of the structured parts on the 269 adopted mesh generation algorithm. It follows that the Hybrid GBL is locally 270 quite similar to the Full ortho GBL in such a region. 271

A quantitative statistical description of the patterns above is given in Table 272 1. Overall, two main comments follow. First, the maximum values of both 273 metrics are comparable among GBLs, and moderate $(\max(\phi) \leq 0.5)$. In other 274 terms, cells in GBLs are moderately skewed (the intersection point is no longer 275 distant than $|F_{I-II}|/4$ from the face mid-point) and moderately non-orthogonal 276 $(\theta \leq 45^{\circ})$. Second, s^* and θ^* values significantly greater than zero $(\phi \geq 0.1)$ 277 rarely occur, except for θ^* in O-type GBL. In other terms, GBL are very locally 278 skewed only. In follows that average and median values are very low, in turn. 279 Bearing in mind that non-orthogonality errors will be numerically corrected, it 280 might be inferred that all GBLs have comparable overall good quality, while 281 point-wise moderately-skewed cells are source of errors only. 282

ϕ	GBL	$\max(\phi)$	$\%_f \phi > 0.1$	$\operatorname{avg}(\phi)$	$\mathrm{median}(\phi)$	discrete distributions
θ^*	Hybrid	0.2857	7.2%	0.0223	0	$#_f/#_T$
	O-type	0.4955	83.9%	0.2778	0.2952	
						1e-2
s^*	Hybrid	0.5532	8.5%	0.0241	1e-6	Ie-3 0 0.1 0.2 0.3 0.4 0.5 s* 0 0.1 0.2 0.3 0.4 0.5 g*
	O-type	0.4884	0.7%	0.0061	0.0047	(a) (b)

Table 1: Statistics on dimensionless skewness and non-orthogonality for different GBL types

283 4. Results

The present results are compared and discussed among them, in order to 284 evaluate the errors induced by skewed cells in different regions of the GBL 285 (Hybrid, O-type and O-type dnw grids) with respect to the full orthogonal 286 grid (Full ortho). Furthermore, the present results are compared with a huge, 287 even if not necessarily exhaustive, number of published studies on the same 288 flow around square cylinder at Re=22,000, in the wake of the Ercoftac QNET-289 CFD Knowledge Base Wiki benchmark (Rodi, 2004, 2012). The goal of such a 290 comparison is twofold: first, assessing the overall modelling error of the GBL-291 error-free simulation (Full ortho) with respect to the most accurate results in 292 literature, and to compare it with the skewness-induced errors; second, placing 293 the present results in the perspective of the whole scientific production on the 294 topic, with special interest to the variability of the results obtained with different 295 approaches. Both experimental tests spanning about 50 years (Pocha, 1971; 296 Wilkinson, 1974; Lee, 1975; Otsuki et al., 1978; Bearman and Obasaju, 1982; 297 Durao et al., 1988; Sakamoto et al., 1989; Lyn and Rodi, 1994; Lyn et al., 1995; 298 Nishimura, 2001; Noda and Nakayama, 2003; Liu et al., 2015; Moore et al., 2019, 299 in chronological order), and computational studies are considered. Among the 300 latter, different approaches to turbulence modelling are considered: from the 301

pioneering so-called No-Model Simulations (NMS, Tamura and Ito, 1996) and
LES (Murakami and Mochida, 1995; Yu and Kareem, 1996), to URANS (Franke
and Rodi, 1991; Kato and Launder, 1993; Deng et al., 1994; Wang and Vanka,
1997; Bosch and Rodi, 1998; Shimada and Ishihara, 2002; Younis and Przulj,
2005) and hybrid models (Ke, 2019), up to LES (Sohankar et al., 2000; Oka and
Ishihara, 2009; Cao and Tamura, 2016; Chen et al., 2020), recent NMS (Cao
et al., 2020), and DNS (Trias et al., 2015).

For the sake of clarity, in the following the obtained results are schematically arranged in three subsections. The simulated flow field around the cylinder is discussed first, the distributions of the stresses at wall are commented in the second section, while the bulk force coefficients are compared in the last one.

313 4.1. Flow field

In order to provide an overall sound phenomenological picture of the sim-314 ulated flow topology, the instantaneous flow patterns around the cylinder and 315 in its wake are plotted in Figure 6. All the flow patterns refer to the same 316 vortex-shedding phase, corresponding to the local maxima of the lift force. For 317 the sake of brevity, the isocontours of the vorticity magnitude $|\omega_z|$ from two 318 grids are included only, corresponding to the highest (Full ortho, Fig. 6-d) and 319 lowest (O-type, Fig. 6-e) grid quality. The present results are compared be-320 tween them and with analogous flow visualizations from past studies: DNS by 321 Trias et al. (2015) (background color according to the magnitude of the pres-322 sure gradient); NMS by Cao et al. (2020) (densest z-wise grid, background color 323 according to the streamwise velocity component): 2D URANS simulation by 324 Shimada and Ishihara (2002) (isocontours of the vorticity magnitude). Visual-325 izations are sorted by expected decreasing overall accuracy of the computational 326 model. In spite of their different state variables and contour values, qualitative 327 and quantitative remarks can be drawn. The von Kármán-like vortex street 328 clearly emerges in the wake in every simulation. However, the simulated vortex 329 pattern exhibits significant differences. A common trend can be recognized in 330 visualizations from a) to d): the lower the expected overall model accuracy, the 331

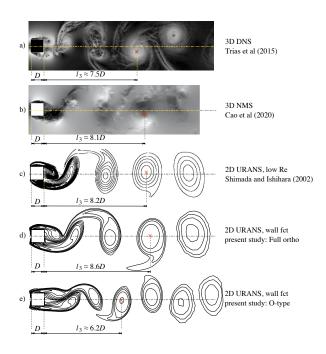


Figure 6: Instantaneous flow fields along the wake at the same vortex-shedding phase $(t = argmax(C_L))$ simulated by different computational models

slightly longer the longitudinal spacing between successive vortices. Conversely, 332 this trend is no longer monotonic moving from the Full ortho (d) to the skewed 333 O-type GBL (e): the vortex spacing drastically shortens. The distance l_3 be-334 tween the trailing edge and the centre of the 3rd vortex in the wake is estimated 335 by hand-picking to quantitatively support such a qualitative trend. Correspond-336 ingly, the wake width simulated by skewed O-type GBL (e) is clearly narrower 337 than in Full ortho GBL (d). The wake features above appear correlated to the 338 curvature of the shear layer along the side surface just downwind the separation 339 point at the leading edge, and to the vortex shedding close to the trailing edge, 340 in turn. Simulations (a)-(d) are qualitatively consistent in predicting a highly 341 curved shear layer, and shedding point close to the rear face. Conversely, skewed 342 O-type GBL returns a lower curvature of the shear layer, and a shedding point 343 further away in the wake. 344

Figure 7 shows the time averaged vorticity $t - avg(\omega_z)$ field around the cylinder base surface. The experimental PIV-based visualization by Moore et al.

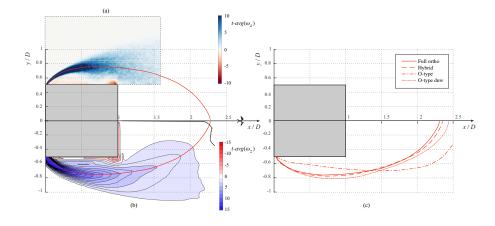


Figure 7: Mean flow field around the cylinder: time averaged vorticity $t - avg(\omega_z)$ measured by Moore et al. (2019) (a), and simulated by the full orthogonal GBL (b); shape of the shear layer by loci of $max(t - avg(\omega_z))$ simulated by different GBLs (c)

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(2019) is given in Figure 7(a), while the corresponding field simulated by the Full 347 ortho GBL is plotted in Figure 7(b). Both fields refer to the lower side surface, 348 the experimental field being mirrored along the x axis to facilitate comparison. 349 The red curve in both subfigures is the locus of maxima $max(t - avg(\omega_z))$ sim-350 ulated by Full ortho GBL, i.e. it visualizes the shape of the simulated shear 351 layer. The Full ortho simulation is in excellent qualitatively agreement with 352 the experiment, in spite of the relatively simple overall computational model. 353 The red curve closely follows the axis of the experimental blue plume, i.e. the 354 separated shear layer. Finally, the Full ortho GBL simulates the negative time-355 averaged vorticity induced by the secondary clockwise vortex at the trailing 356 edge, in qualitative agreement with PIV measurements. Figure 7(c) collects 357 the loci of $max(t - avg(\omega_z))$ for the four different GBLs. It can be observed 358 that: i. the shapes of the shear layer obtained by the Full ortho and the Hybrid 359 GBLs are very close in spite of the scatted skewness shown by the latter; ii. 360 the O-type dnw curve slightly differs from the previous two, especially in the 361

wake; iii. the O-type curve completely departs from the others. On the one 362 hand, such findings quantitatively confirm in time average what discussed in 363 qualitative terms about the instantaneous patterns (Fig. 6-d, -e). On the other 364 hand, they suggest that very local skewness-induced errors along the diagonals 365 of the square annular partition globally affect the whole separated flow. The 366 closer the error source to the separation point at the cylinder edge, the more 367 significant its global effects (see e.g. the O-type GBL in Fig. 5-d). Conversely, 368 a local high quality of the cell closely around the separation point preserves the 369 accuracy of the results in spite of high skewness elsewhere (see e.g. the Hybrid 370 GBL in Fig. 5-d). Such errors are introduced along the diagonals of the square 371 annular partition, and in particular close to the separation point at the cylinder 372 edge in the O-type GBL. 373

In order to check such a conjecture, Figure 8 relates the skewness viscosity to other simulated flow variables in the lower near wall region. The time

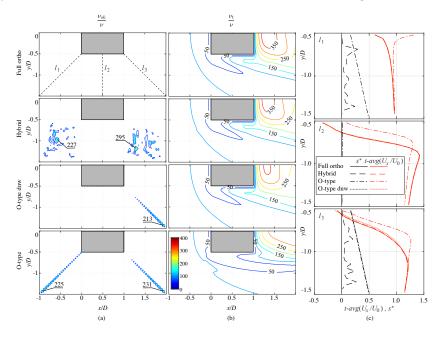


Figure 8: Time-averaged skewness viscosity ratio (a), turbulent viscosity ratio (b), x-velocity (red lines) and skewness (black lines) profiles (c)

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averaged skewness viscosity (ν_{sk} , Eq. 6, Fig. 8-a) are compared to the time av-376 eraged turbulence eddy viscosity of the URANS model (ν_t , Eq.s 3-4, Fig. 8-b). 377 Both quantities are normalized with respect to the kinematic viscosity ν . Two 378 main remarks follow. First, ν_{sk} is not distributed all over the domain but only 379 where skewed faces are located (Fig. 5-d). Second, ν_{sk} and ν_t take on values 380 within the same order of magnitude, hundreds of time larger than the kinematic 381 viscosity: in other term, false skewness viscosity potentially affects the solution 382 to the same extent of the turbulence model. In particular, ν_{sk} has a dramatic 383 relative importance upwind the leading edge, where transition to turbulence not 384 yet occurs, and ν_t is very low. Three different 1D domains l_1, l_2, l_3 are defined, 385 as sketched in Figure 8(a), first row. Figure 8(c) shows the profiles of the di-386 mensionless skewness and time-averaged horizontal velocity U_x along such lines. 387 Three main remarks follow. First, skewed faces lie along l_1 in both Hybrid and 388 O-type GBL. However, only the O-type U_x profile departs from the others and 389 shows a non-physical flow acceleration very close to the wall. We can infer that 390 skewness viscosity plays a dramatic detrimental role mainly close to the wall, 391 approximately in the first 2 / 4 cell rows (see Fig. 5-d). Second, an analogous 392 scatter between O-type GBL and the other simulations is observed along l_2 , 393 even if the cells are locally unskewed and orthogonal in all GBLs. It means that 394 the local skewness error introduced by O-type GBL along l_1 is transported by 395 convection along the side surface, analogously to the modeled state variables. 396 From a physical point of view, this implies *global effects*: the reversed flow pre-397 dicted by Full ortho, Hybrid and O-type dnw GBLs is not adequately grasped 398 by the O-type GBL. Finally, the very small scatter between the U_x profiles by 399 Full ortho and O-type dnw GBLs along l_3 testifies that the skewness viscosity, 400 its error and its effects not depend only on s^* , but on the velocity field, too (Eq. 401 **6**). 402

In order to quantitatively compare the present results with the ones in literature, the distributions of the dimensionless streamwise velocity U_x/U_0 averaged in time (t-avg in the following) and in the spanwise direction (z-avg in the following, relevant to 3D models) are plotted in Figure 9(a) along the wake centerline y = 0. In particular, available experimental measurements (Durao
et al., 1988; Lyn and Rodi, 1994), high-fidelity DNS (Trias et al., 2015) and
LES (Cao and Tamura, 2016), and other LES and URANS results are considered. In general, the considered high-fidelity simulations provide consistent

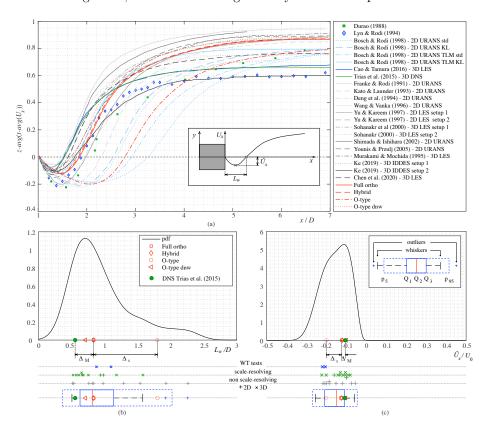


Figure 9: Distribution of time-averaged x-velocity along y = 0 (a), statistics of the wake recirculation length L_w (b) and velocity defect \breve{U}_x (c)

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results. However, as pointed out by the same Trias et al. (2015), their computational results slightly but systematically differ from the reference experimental data of Lyn and Rodi (1994), with remarkable differences in the near wake recirculation region, where the simulated reversed velocity is nearly half of the measured one. The Lyn and Rodi (1994) measurements significantly depart in turn from the previous measures by Durao et al. (1988), and even larger variabil-

ity takes place among other computational results, even if the same approach to 417 turbulence modelling is adopted. Such overall variability of the velocity profiles 418 results in scattered values of the recirculation length, of the maximum reversed 419 speed, and of the velocity plateau along the fare wake, if any. In particular, 420 the streamwise velocity profiles predicted by low-skewness grids (Full ortho and 421 Hybrid) are in satisfactory agreement with experimental data in the near wake, 422 while the velocity recovery is overestimated in the far wake x/D > 2.5. The 423 dense and orthogonal grid in the wake (Fig. 5), and the 3rd order scheme used 424 for the convective terms suggest that most of the underlying error can be tenta-425 tively ascribed to the 2D domain, and/or to adopted turbulence model, namely 426 to the empirical modelling of $C^*_{\varepsilon_2}$. The velocity distribution issued from O-type 427 grid dramatically suffers the effects of the skewness viscosity (eq. 6), resulting 428 in the striking overestimation of the recirculation length. Conversely, skewness 429 around the trailing edge only (O-type dnw) has the opposite effects of the skew-430 ness around both leading and trailing edges (O-type), i.e. it shortens the recircu-431 lation length and reduces the velocity defect with respect to the Full ortho grid. 432 Two local metrics are retained in order to synthetically describe the near wake 433 recirculation region: the wake recirculation length L_w , and the velocity defect 434 \check{U}_x , graphically defined in Figure 9(a) for the sake of clarity and conciseness. In 435 the light of the high variability highlighted above, statistics are obtained on each 436 of them, by referring to the ensemble of realizations collected in literature and 437 plotted in Figure 9(a). Statistics are visualized in Figures 9(b) and (c) for L_w/D 438 and U_x/U_0 , respectively. All realizations are reported and categorised in Wind 439 Tunnel (WT) measurements, non scale-resolving (RANS/URANS), and scale-440 resolving (DNS, NMS, LES, DES, IDDES) simulations. The *a-priori* expected 441 highest-fidelity model (DNS, Trias et al., 2015) is pointed out. Non paramet-442 rical Probability Density Functions (PDF) with positive and negative support 443 are fitted on the whole ensemble for L_w and \check{U}_x , respectively, and accompanied 444 by the corresponding box plots. Such a postprocessing allows a twofold reading 445 of the obtained results. In a deterministic reading, the present overall mod-446 elling error Δ_M is evaluated by the scatter between the highest-fidelity model 447

(DNS, Trias et al., 2015) and the present highest-quality GBL (Full ortho), 448 while the highest skewness-induced error Δ_s is given by the scatter between the 449 highest (Full ortho) and lowest (O-type) GBL quality. In a purely statistical 450 reading, each realization can be associated with the corresponding percentile, 451 i.e. the number where a certain percentage of scores fall below it. Here, we are 452 aware that the ensemble cardinality (#=22) is significant but not necessarily 453 high enough to ensure full statistical convergence. Hence, we prefer a coarse 454 statistical mapping of the present results by referring to outliers, and to the 455 intervals between percentiles p_5 , $p_{25} = Q_1$, $p_{50} = Q_2$, $p_{75} = Q_3$, p_{95} . Selected 456 remarks follow from both perspectives: i. for both metrics the skewness-induced 457 error is by far larger than the overall modelling error, despite the latter results 458 form top- (3D DNS) and low- (2D URANS) ranked models; ii. L_w/D shows a 459 variability (coefficient of variation $c.o.v.(L_w) = \sigma(L_w)/|\mu(L_w)| = 0.236$) lower 460 than \check{U}_x/U_0 (c.o.v. $(\check{U}_x) = 0.364$); iii. the values provided by the Full ortho and 461 Hybrid grids are within the $[Q_1Q_3]$ range for both metrics, while the L_w/D 462 value from O-type grid is clearly an outlier, and the corresponding \check{U}_x/U_0 is 463 close to Q_1 . 464

465 4.2. Stress distributions at wall

The distributions of the dimensionless wall shear stress τ_w averaged in time 466 and in the spanwise direction are plotted in Figure 10 versus the curvilinear 467 axis s along the upper half perimeter. Positive values correspond to τ_w with the 468 same direction of the curvilinear axis. The distributions obtained by different 469 GBLs are compared with the ones obtained by Cao and Tamura (2016) with 470 different grid densities. A closed view along the side surface is given, in anal-471 ogy to what done by Cao and Tamura (2016) as well. The distributions by 472 this study follows from a wall-function modelling approach and, consequently, 473 the size of the cells along the wall is much larger than the ones in Cao and 474 Tamura (2016) in both s and n directions. The s-wise low grid density implies 475 a scarce resolution of the wall shear stress profile, testified by the coarse place-476 ment of the peaks immediately upstream and downstream the leading corner B. 477

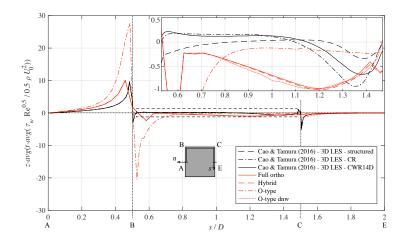


Figure 10: Distribution of time-averaged dimensionless wall shear stress along the upper half perimeter

However, the maxima of $|\tau_w \text{Re}^{0.5}/0.5\rho U_0^2| \approx 10$ and 2.5 predicted by Full ortho, 478 Hybrid and O-type dnw GBLs well matches the LES results in Cao and Tamura 479 (2016). Both s and n-wise low grid density does not allows the simulation of the 480 small-size counterclockwise secondary vortices downstream B ($0.5 < s/D \lesssim 0.7$) 481 and upstream the trailing corner C (1.47 $\lesssim s/D < 1.5)$ visualized in Cao and 482 Tamura (2016)-Fig. 10, and Cao et al. (2020)-Fig. 10. This yields to reversed 483 flow all along the side surface, and corresponding negative τ_w . The "3D LES-484 structured" simulation by Cao and Tamura (2016) is affected by an analogous 485 issue. The O-type distribution drastically differs from the others because of both 486 local and global effects of the skewness viscosity. The local peaks around B are, 487 in absolute value, from 3 to 8 times higher than the ones issued by the other 488 GBLs. After a shear recovery, the distribution attains a plateau value close to 489 zero along the downwind half of the side surface, i.e. the flow is unrealistically 490 quasi-stagnant. Conversely, the effects of the skewness viscosity on τ_w induced 491 by O-type dnw GBL mainly take place around the trailing corner only, because 492 of the convection-dominated flow. 493



The distributions of the time-averaged and time-standard deviation (t - std

in the following) of the pressure coefficient Cp, also averaged spanwise and be-495 tween the upper and lower half perimeters (side -avg in the following) are 496 plotted in Figure 11 (a) and (b), respectively. They are compared to available 497 experimental measurements, high-fidelity DNS (Trias et al., 2015) and LES 498 (Cao and Tamura, 2016), and several other LES and URANS simulations. In 499 spite of the same nominal setup, both experimental and computational results 500 are significantly scattered along the whole base surface, especially in regard to 501 t - std(Cp). Interestingly, distribution from *a-priori* high-fidelity simulations 502 are extremal occurrences at the lower and upper bounds of the ensemble for 503 -avq(Cp) and t-std(Cp), respectively. Conversely, Full ortho GBL distribu-504 t tions lie around the upper and lower bounds of the ensemble for t - avq(Cp) and 505 - std(Cp), respectively. In short, the adopted computational model slightly t 506 underestimates the mean suction and the pressure fluctuations, and notably 507 fails in predicting the local maximum of t - std(Cp) in the downwind half of the 508 side surface. Hybrid GBL provides the same mean distribution, and negligible 509 differences take place for the pressure standard deviation. O-type dnw GBL 510 does not significantly affect the mean distribution, while the local error made at 511 the trailing edge C makes the t - std(Cp) distribution closer to the average of 512 the experimental measurements. We conjecture that such a misleading agree-513 ment is induced by a deeper footprint of the vortex shedding on the side surface, 514 induced by small changes in the flow structure in the near wake. Highly skewed 515 O-type GBL dramatically and qualitatively affects both the distributions of the 516 Cp mean and standard deviation. Their trend is typical of a streamlined body 517 rather than of a bluff body: deep mean suction occurs at the leading edge B; 518 mean pressure plateau is missing downstream it, and mean pressure recovery 519 takes place along the side surface; pressure standard deviation is one order of 520 magnitude lower than the average of the results in literature. 521

Analogously to what done for the streamwise velocity in the wake, four swise averaged metrics are defined to synthetically describe the time-averaged and time-fluctuating pressure field along the side (l - avg) and rear (b - avg)surfaces. The bounds of the averaging lengths l1, l2, b are intended to discard

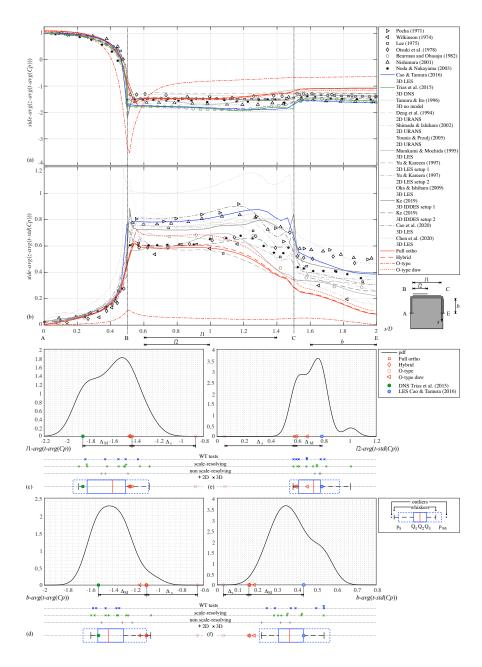


Figure 11: Cp distribution: time-averaged value (a), time-standard deviation (b), statistics of the side (c,e) and base (d,f) pressure

the neighbourhood of the sharp edges, and related local high pressure gradients. 526 The cardinality of the ensemble of the realizations (#=21) is close to the one 527 used for the streamwise velocity in the wake. The 3D LES by Cao and Tamura 528 (2016) is chosen as *a-priori* highest-fidelity model when DNS results from Trias 529 et al. (2015) are not publicly available. Figures 11(c) and (d) show the distribu-530 tions of the side and rear surface t - avq(Cp) plateaux. The following remarks 531 can be drawn: i. the coefficients of variation of the literature results are similar 532 (c.o.v.(l1 - avg(t - avg(Cp))) = 0.12, c.o.v.(b - avg(t - avg(Cp))) = 0.1) and 533 not negligible; ii. in a statistical setting, the Full ortho, Hybrid and O-type 534 dnw realizations are close to Q_3 of l1 - avg(t - avg(Cp)) and to and p_{95} of 535 b - avg(t - avg(Cp)), while the O-type ones are outliers in both cases; iii. in a 536 deterministic setting, the skewness errors Δ_s are comparable to the model ones 537 Δ_M . Figures 11(e) and (f) show the distributions of t - std(Cp) over l2 and b, 538 respectively. The following remarks can be drawn: i. the c.o.v. of the distribu-539 tions of the Cp standard deviation in literature are overall higher than the ones 540 of the time-averaged pressure. In particular, the c.o.v. of the pressure standard 541 deviation along the rear face (c.o.v.(b - avg(t - std(Cp))) = 0.25, Fig. 11-f) is 542 higher than the one on the side surface (c.o.v.(l2 - avg(t - std(Cp))) = 0.15, Fig. 543 11-e), and impressive in absolute terms. The slight bi-modality of the PDF in 544 Figure 11(e) does not seem to depend on the approach/model used; ii. over the 545 l_2 domain, while the Full ortho and Hybrid realizations are located near Q_1 and 546 the O-type dnw one in between Q_1 and Q_2 , the O-type realization is an outlier; 547 iii. over the domain b, all the realizations of the present work are outliers even if 548 Full ortho, Hybrid and O-type dnw are close to p_5 ; iv. the deterministic model 549 error Δ_M is prevailing over the skewness one Δ_s along b only, while the opposite 550 is the case along l2. 551

552 4.3. Bulk forces

⁵⁵³ Bulk force coefficients are scarcely useful in shedding light on phenomenolog-⁵⁵⁴ ical local features of the simulated flows and skewness effects on them. Neverthe-⁵⁵⁵ less, they are of great importance for design purposes in engineering applications.

The main bulk force coefficients obtained in the present study are compared and 556 discussed with reference to statistics on the same metrics obtained in different 557 wind tunnel and computational studies in Figure 12: $t - avg(C_D)$ is the time-558 and spanwise-averaged drag coefficient per unit length; $t - std(C_L)$ is the stan-559 dard deviation of the time variation of the lift coefficient; $St = f_s D/U_0$ is the 560 Strouhal number, where the shedding frequency f_s is evaluated from the time 561 fluctuations of the lift coefficient. Statistics are obtained from ensembles having 562 a cardinality equal to #=39 for $t - avg(C_D)$ and St, and #=29 for $t - std(C_L)$. 563 Realizations are collected from Liu et al. (2015), Trias et al. (2015), Cao and 564 Tamura (2016), and references therein. PDFs of $t - avg(C_D)$ and $t - std(C_L)$

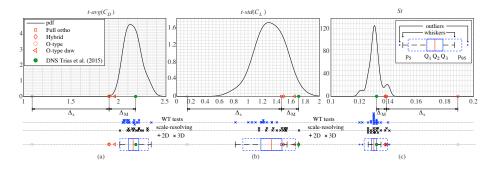


Figure 12: Bulk parameters: statistics of time-averaged drag coefficient (a), standard deviation of the lift coefficient (b), Strouhal number (c)

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are moderately skewed and unimodal. The three-mode PDF of the St number 566 seems not due to the selected approach: for instance, WT realizations contribute 567 to all modes. The coefficient of variation is small for the time-averaged drag 568 $(c.o.v.(t - avg(C_D)) = 0.04)$ and Strouhal number (c.o.v.(St) = 0.03), while 569 it is significant for lift fluctuations $(c.o.v.(t - std(C_L)) = 0.15)$. The *a priori* 570 high fidelity model closely fits the ensemble average value of $t - avg(C_D)$ and 571 St. Conversely, DNS significantly overestimates the $t - std(C_L)$ in respect to 572 the whole ensemble, and to the experimental measurements in particular. We 573 conjecture this can be ascribed to the unsatisfactory simulation of the spanwise 574 pressure correlation, probably due to too short spanwise domain size, and/or to 575

poor grid density in the same direction (Bruno et al., 2012). Both modelling and 576 skewness errors imply the reduction of $t - avg(C_D)$ and $t - std(C_L)$, while the 577 Strouhal number increases under the effects of both Δ_M and Δ_S . The skewness 578 error is much higher than the modelling one for all bulk metrics: 2.9 times for 579 $t - avg(C_D)$, 5.4 times for $t - std(C_L)$, 7.2 times for St. Let us consider the 580 drag coefficient as an example (Fig. 12-a). The ensemble median is $Q_2 \approx 2.15$, 581 and DNS returns a reference value $t - avg(C_D)_{rep} \approx 2.18$ (71-th percentile in in 582 probabilistic terms), while Full ortho and Hybrid GBL predict $t - avg(C_D) \approx 1.9$ 583 (corresponding to the 0.01-th percentile, and resulting in $\Delta_M \approx 0.28$), and O-584 type GBL $t - avg(C_D) \approx 1.1$ (corresponding to a full outlier, and resulting in 585 $\Delta_s \approx 1.08$). It follows that both modelling and skewness errors lead to the 586 unsafe estimation of the drag force. However, the design could be effectively 587 secured versus Δ_M by adopting a partial safety factor γ_Q for variable actions, 588 also accounting for model errors/uncertainties (e.g. $\gamma_Q = 1.5$ in EN 1990, 2002; 589 EN 1991-1-4, 2005, for wind action in structural engineering). The same usual 590 value of the partial safety factor is not effective face to the large skewness error. 591 Analogous comments are relevant to $t - std(C_L)$ and St, if the deviation of the 592 Strouhal number has unfavourable effects on the structural response. 593

594 5. Conclusions

The flow around a square cylinder at Re=22,000 is simulated under four 595 types of Grid Boundary Layers, the other components of the computational 596 model being unchanged. GBLs differ in cell quality, while they share common 597 other features, such as the geometry of their boundaries, their size, the grid den-598 sity at wall, the grid in the outer part of the computational domain. Thanks to 599 this, the effects of the cell skewness are separately discussed with respect to the 600 ideal full orthogonal GBL. The attention paid in the recent past to the adopted 601 602 benchmark by the scientific community allowed a huge number of previous results to be collected and statistically analyzed. Hence, the obtained results are 603 compared not only among them, but also with such a huge database. 604

⁶⁰⁵ Some conclusions can be summarized:

• The local error induced by few, moderately skewed, near-wall cells upwind the cylinder propagates windward because of the convection-dominated problem. The error globally and significantly affects the boundary layer separation and the vortex shedding in the wake. The same skewed cells around the trailing edge only affect the flow to a lower extent;

The skewness error on bulk aerodynamic coefficients may largely prevails
 on the overall modelling error, despite a very simple turbulence model and
 a second-order accurate numerical approach are deliberately adopted in
 the study;

O-type, block structured, skewed GBL results in the qualitative simulation of the instantaneous vortex street in the wake, and in the concurrent dramatic underestimation of the lift fluctuating component by a factor
Such results call to mind the well renowned quote by Ferziger (1993): "The greatest disaster one can encounter in computation is not instability or lack of convergence, but results that are simultaneously good enough to be believable, but bad enough to cause trouble";

• Hybrid GBL with unstructured patterns around the cylinder edges can provide results analogous to the ones obtained with a full orthogonal grid, provided that from 2 to 4 cell rows nearly orthogonal and skewness-free are generated near the wall (corresponding to a width from 1/12D to 1/6D), while clusters of skewed cells are kept further from the wall;

Hybrid GBL is judged to be a fine balance between accuracy and flexibility
 in grid generation, with the above due care and attention to detail. In
 particular, hybrid GBLs are nearly unescapable for real-world engineering
 applications with intricate geometries, e.g. with multiple obtuse or acute
 edges.

Analogous studies on grid quality effects are recommended for other paradig matic high-Re flows in bluff body aerodynamics, such as the one around elon-

gated rectangular cylinders with separated and reattached boundary layer, and
around real world structures, such as bridge decks and tall buildings. The relative significance of skewness viscosity with respect to other modelling approaches
to turbulence such as wall resolved LES would be worth to be investigated as
well.

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645 References

- Ahipo, Y., Traoré, P., 2009. A robust iterative scheme for finite volume discretization of diffusive flux on highly skewed meshes. Journal of Computational and Applied Mathematics 231, 478–491. doi:10.1016/j.cam.2009.03.
 015.
- Antepara, O., Lehmkuhl, O., Borrell, R., Chiva, J., Oliva, A., 2015. Parallel adaptive mesh refinement for large-eddy simulations of turbulent flows.
 Computers and Fluids 110, 48–61. doi:10.1016/j.compfluid.2014.09.050.
- Bearman, P.W., Obasaju, E.D., 1982. An experimental study of pressure fluctuations on fixed and oscillating square-section cylinders. Journal of Fluid
 Mechanics 119, 297–321. doi:10.1017/s0022112082001360.
- Berrone, S., Marro, M., 2009. Space-time adaptive simulations for unsteady
 navier-stokes problems. Computers and Fluids 38, 1132-1144. doi:10.1016/
 j.compfluid.2008.11.004.

Blacker, T., Stephenson, M., 1991. Paving: A new approach to automated
quadrilateral mesh generation. International Journal for Numerical Methods
in Engineering 32, 811–847. doi:10.1002/nme.1620320410.

- Bosch, G., Rodi, W., 1998. Simulation of vortex shedding past a square cylinder
 with different turbulence models. International Journal for Numerical Methods in Fluids 28, 601–616. doi:10.1002/(sici)1097-0363(19980930)28:
 4<601::aid-fld732>3.0.co;2-f.
- Bruno, L., Coste, N., Fransos, D., 2012. Simulated flow around a rectangular 5:1
 cylinder: Spanwise discretisation effects and emerging flow features. Journal
 of Wind Engineering and Industrial Aerodynamics 104-106. doi:10.1016/j.
 jweia.2012.03.018.
- Bruno, L., Salvetti, M., Ricciardelli, F., 2014. Benchmark on the aerodynamics
 of a rectangular 5:1 cylinder: An overview after the first four years of activity.
 Journal of Wind Engineering and Industrial Aerodynamics 126. doi:10.1016/
 j.jweia.2014.01.005.
- Bruno, L., Salvetti, M.V., 2017. Ufr 2-15, benchmark on the aerodynamics of a
 rectangular 5:1 cylinder (barc). URL: https://www.kbwiki.ercoftac.org/
 w/index.php/UFR_2-15_Description.
- Cao, Y., Tamura, T., 2016. Large-eddy simulations of flow past a square cylinder
 using structured and unstructured grids. Computers and Fluids 137, 36–54.
 doi:10.1016/j.compfluid.2016.07.013.
- Cao, Y., Tamura, T., 2017. Supercritical flows past a square cylinder with
 rounded corners. Physics of Fluids 29, 085110. doi:10.1063/1.4998739.
- Cao, Y., Tamura, T., 2018. Aerodynamic characteristics of a rounded-corner
 square cylinder in shear flow at subcritical and supercritical revnolds numbers.
- Journal of Fluids and Structures 82, 473-491. doi:https://doi.org/10.
- 685 1016/j.jfluidstructs.2018.07.012.

- Cao, Y., Tamura, T., Kawai, H., 2020. Spanwise resolution requirements for the 686
- simulation of high-reynolds-number flows past a square cylinder. Computers 687
- & Fluids 196, 104320. doi:10.1016/j.compfluid.2019.104320. 688
- Chen, Y., Djidjeli, K., Xie, Z.T., 2020. Large eddy simulation of flow past 689 stationary and oscillating square cylinders. Journal of Fluids and Structures 690 97, 103107. doi:10.1016/j.jfluidstructs.2020.103107. 691
- Deng, G., Piquet, J., Queutey, P., Visonneau, M., 1994. 2-d computations of 692 unsteady flow past a square cylinder with the baldwin-lomax model. Journal 693 of Fluids and Structures 8, 663-680. doi:10.1016/s0889-9746(94)90140-6. 694
- Denner, F., van Wachem, B., 2014. Compressive vof method with skewness 695 correction to capture sharp interfaces on arbitrary meshes. Journal of Com-696 putational Physics 279, 127–144. doi:10.1016/j.jcp.2014.09.002. 697
- Denner, F., van Wachem, B., 2015. Tvd differencing on three-dimensional un-698 structured meshes with monotonicity-preserving correction of mesh skewness. 699 Journal of Computational Physics 298, 466–479. doi:10.1016/j.jcp.2015. 700 06.008.
- Durao, D.F.G., Heitor, M.V., Pereira, J.C.F., 1988. Measurements of turbulent 702 and periodic flows around a square cross-section cylinder. Experiments in 703
 - Fluids 6, 298-304. doi:10.1007/bf00538820. 704

701

- EN 1990, 2002. Eurocode basis of structural design. 705
- EN 1991-1-4, 2005. Eurocode 1: Actions on structures part 1-4: General 706 actions - wind actions. 707
- Ferziger, J.H., 1993. Estimation and reduction of numerical error, in: Proceed-708
- ings of the Symposium on Quantification of Uncertainty in Computational 709 Fluid Dynamics, pp. 1–8. 710
- Franke, R., Rodi, W., 1991. Calculation of vortex shedding past a square 711 cylinder with various turbulence models, in: Eighth Symposium on 1urbu-712

- lent Shear Flows, Paper 20-1, Technical University of Munich, Germany.
 doi:10.1007/978-3-642-77674-8_14.
- Haque, M., Katsuchi, H., Yamada, H., Nishio, M., 2016. Strategy to develop efficient grid system for flow analysis around two-dimensional bluff
 bodies. KSCE Journal of Civil Engineering 20, 1913–1924. doi:10.1007/
 \$12205-015-0696-2.
- Hoffman, J., 2005. Computation of mean drag for bluff body problems using
 adaptive dns/les. SIAM Journal on Scientific Computing 27, 184–207. doi:10.
 1137/040614463.
- Issa, R.I., 1986. Solution of implicitly discretized fluid-flow equations by operator splitting. Journal of Computational Physics 62, 40–65. doi:10.1016/
 0021-9991(86)90099-9.
- Jasak, H., 1996. Error Analysis and Estimation for the Finite Volume Method
 with Applications to Fluid Flows. Ph.D. thesis.
- Juretić, F., Gosman, A.D., 2010. Error analysis of the finite-volume method
 with respect to mesh type. Numerical Heat Transfer, Part B: Fundamentals
 57, 414–439. doi:10.1080/10407791003685155.
- Kato, M., Launder, B.E., 1993. The modeling of turbulent flow around stationary and vibrating square cylinders, in: Ninth Symposium on Turbulent Shear
 Flows, Paper 10-4, Kyoto, Japan.
- Ke, J., 2019. RANS and hybrid LES/RANS simulations of flow over a square
 cylinder. Advances in Aerodynamics 1. doi:10.1186/s42774-019-0012-9.
- ⁷³⁵ Khosla, P., Rubin, S., 1974. A diagonally dominant second-order accu⁷³⁶ rate implicit scheme. Computers and Fluids 2, 207–209. doi:10.1016/
 ⁷³⁷ 0045-7930(74)90014-0.
- Launder, B., Spalding, D., 1974. The numerical computation of turbulent
 flows. Computer Methods in Applied Mechanics and Engineering 3, 269–289.
 doi:10.1016/b978-0-08-030937-8.50016-7.

Lee, B.E., 1975. The effect of turbulence on the surface pressure field of
a square prism. Journal of Fluid Mechanics 69, 263–282. doi:10.1017/
s0022112075001437.

- Lee, N., Lee, H., Baek, C., Lee, S., 2016. Aeroelastic analysis of bridge deck
 flutter with modified implicit coupling method. Journal of Wind Engineering
 and Industrial Aerodynamics 155, 11–22. doi:10.1016/j.jweia.2016.04.
 010.
- ⁷⁴⁸ Lee, S., 1997. Unsteady aerodynamic force prediction on a square cylinder ⁷⁴⁹ using $k - \varepsilon$ turbulence models. Journal of Wind Engineering and Industrial ⁷⁵⁰ Aerodynamics 67-68, 79–90. doi:10.1016/s0167-6105(97)00064-0.
- Leonard, B., 1979. A stable and accurate convective modelling procedure based
 on quadratic upstream interpolation. Computer Methods in Applied Mechan ics and Engineering 19, 59–98. doi:10.1016/0045-7825(79)90034-3.
- Liu, M., Xiao, L., Yang, L., 2015. Experimental investigation of flow characteristics around four square-cylinder arrays at subcritical reynolds numbers.
 International Journal of Naval Architecture and Ocean Engineering 7, 906–919. doi:10.1515/ijnaoe-2015-0063.
- Lyn, D.A., Einav, S., Rodi, W., Park, J.H., 1995. A laser-doppler velocimetry study of ensemble-averaged characteristics of the turbulent wake
 of a square cylinder. Journal of Fluid Mechanics 304, 285–319. doi:doi:
 10.1017/S0022112095004435.
- Lyn, D.A., Rodi, W., 1994. The flapping shear layer formed by flow separation
 from the forward corner of a square cylinder. Journal of Fluid Mechanics 267,
 353. doi:10.1017/s0022112094001217.
- Mannini, C., Sbragi, G., Schewe, G., 2016. Analysis of self-excited forces for a
 box-girder bridge deck through unsteady rans simulations. Journal of Fluids
 and Structures 63, 57–76. doi:10.1016/j.jfluidstructs.2016.02.007.

- ⁷⁶⁸ Mannini, C., Šoda, A., Schewe, G., 2010. Unsteady rans modelling of flow past
- a rectangular cylinder: Investigation of reynolds number effects. Computers
 and Fluids doi:10.1016/j.compfluid.2010.05.014.
- Mariotti, A., Salvetti, M.V., Omrani, P.S., Witteveen, J.A., 2016. Stochastic
 analysis of the impact of freestream conditions on the aerodynamics of a
 rectangular 5:1 cylinder. Computers and Fluids doi:10.1016/j.compfluid.
- 774 2016.06.008.
- Moore, D.M., Letchford, C.W., Amitay, M., 2019. Energetic scales in a bluff
 body shear layer. Journal of Fluid Mechanics doi:10.1017/jfm.2019.480.
- Murakami, W., Mochida, A., 1995. On turbulent vortex shedding past 2d square
 cylinder predicted by cfd. Journal of Wind Engineering and Industrial Aerodynamics 54-55, 191–211. doi:10.1016/0167-6105(94)00043-D.
- Muzaferija, S., 1994. Adaptative finite volume method for flow prediction using
 unstructured meshes and multigrid approach. Ph.D. thesis.
- Nakane, I., 2013. Effectiveness of grid orthogonality on body fitted curvilinear
 coordinate system in cfd. International Journal of Environmental Science and
 Development 4. doi:10.7763/ijesd.2013.v4.329.
- Nieto, F., Owen, J.S., Hargreaves, D.M., Hernandez, S., 2015. Bridge deck
 flutter derivatives: Efficient numerical evaluation exploiting their interdependence. Journal of Wind Engineering and Industrial Aerodynamics 136,
 138–150. doi:10.1016/j.jweia.2014.11.006.
- Nishimura, H., 2001. Fundamental study of bluff body aerodynamics (in
 Japanese). Ph.D. thesis.
- ⁷⁹¹ Noda, H., Nakayama, A., 2003. Free-stream turbulence effects on the instanta-
- ⁷⁹² neous pressure and forces on cylinders of rectangular cross section. Experi-
- ⁷⁹³ ments in Fluids 34, 332–344. doi:10.1007/s00348-002-0562-0.

Oka, S., Ishihara, T., 2009. Numerical study of aerodynamic characteristics of a
square prism in a uniform flow. Journal of Wind Engineering and Industrial
Aerodynamics 97, 548-559. doi:10.1016/j.jweia.2009.08.006.

de Oliveira Samel Moraes, A., da Cunha Lage, P., Cunha, G., Silva, L., 2013.
Analysis of the non-ortogonality correction of finite volume discretization on
unstructured meshes,, in: 22nd International Congress of Mechanical Engineering (COBEM 2013).

- Otsuki, Y., Fuji, K., Washizu, K., Ohya, A., 1978. Wind tunnel experiments
 on aerodynamic forces and pressure distributions of rectangular cylinders in
 a uniform flow, in: Proceedings of the Fifth Symposium on Wind Effects on
 Structures, pp. 169–176.
- Pocha, J.J., 1971. On unsteady flow past cylinders of square cross-section. Ph.D.
 thesis.
- Riberio, A.F.P., 2011. Unsteady rans modelling of flow past a rectangular
 5:1 cylinder: investigation of edge sharpness effects. Proceedings of the
 Thirteenth International Conference on Wind Engineering, Amsterdam, The
 Netherlands .
- Rocchio, B., Mariotti, A., Salvetti, M., 2020. Flow around a 5:1 rectangular
 cylinder: Effects of upstream-edge rounding. Journal of Wind Engineering &
 Industrial Aerodynamics 204, 104237. doi:10.1016/j.jweia.2020.104237.
- Rodi, W., 1997. Comparison of les and rans calculations of the flow around
 bluff bodies. Journal of Wind Engineering and Industrial Aerodynamics 6971, 55–75. doi:10.1016/s0167-6105(97)00147-5.
- Rodi, W., 2004. Ufr 2-02, flow past cylinder. URL: https://www.kbwiki.
 ercoftac.org/w/index.php?title=UFR_2-02_Description.
- Rodi, W., 2012. The ERCOFTAC knowledge base wiki an aid for validating
 CFD models, in: Process and Plant Safety. Wiley-VCH Verlag GmbH & Co.
- KGaA, pp. 179–187. doi:10.1002/9783527645725.ch11.

Rodi, W., Ferziger, J.H., Breuer, M., Pourquié, M., 1997. Status of large-eddy 822

simulation: Results of a workshop. Journal of Fluids Engineering 119, 248-823 262. doi:10.1115/1.2819128. 824

Sakamoto, H., Haniu, H., Kobayashi, Y., 1989. Fluctuating force acting on rect-825 angular cylinders in uniform fow on rectangular cylinders with fully separated 826 flow. Transactions of the Japan Society of Mechanical Engineers, Series B 55, 827 2310 - 2317.828

Schewe, G., 2013. Reynolds-number-effects in flow around a rectangular cylinder 829 with aspect ratio 1:5. Journal of Fluids and Structures 39, 15-26. doi:10. 830 1016/j.jfluidstructs.2013.02.013. 831

Scruton, C., Rogers, E., Menzies, J., Scorer, R., 1971. Steady and unsteady wind 832 loading of buildings and structures [and discussion]. Philosophical Transac-833 tions of the Royal Society of London Series A, Mathematical and Physical 834 Sciences 269, 353-383. URL: https://www.jstor.org/stable/73873. 835

Shimada, K., Ishihara, T., 2002. Application of a modified $k-\varepsilon$ model to the pre-836 diction of aerodynamic characteristics of rectangular cross-section cylinders. 837 Journal of Fluids and Structures 16, 465-485. doi:10.1006/jfls.2001.0433. 838

Shyy, W., Thakur, S., Wright, J., 1992. Second-order upwind and central differ-839 ence schemes for recirculating flow computation. AIAA Journal 4, 923–932. 840

doi:10.2514/3.11010. 841

847

Sohankar, A., Davidson, L., Norberg, C., 2000. Large eddy simulation of flow 842 past a square cylinder: Comparison of different subgrid scale models. Journal 843 of Fluids Engineering 122, 39-47. doi:10.1115/1.483224. 844

Tamura, T., Ito, Y., 1996. Aerodynamic characteristics and flow structures 845 around a rectangular cylinder with a section of various depth/breadth ratios. 846

Journal of Structural and Construction Engineering (Transactions of Archi-

tectural Institute of Japan) 486, 153-162. doi:10.3130/aijs.61.153_2. 848

Tamura, T., Miyagi, T., Kitagishi, T., 1998. Numerical prediction of un-849 steady pressures on a square cylinder with various corner shapes. Journal 850 of Wind Engineering and Industrial Aerodynamics 74-76, 531-542. doi:10. 851 1016/s0167-6105(98)00048-8. 852

- Tang, H., Shum, K., Li, Y., 2019. Investigation of flutter performance of a twin-853 box bridge girder at large angles of attack. Journal of Wind Engineering and 854 Industrial Aerodynamics 186, 192-203. doi:10.1016/j.jweia.2019.01.010. 855
- Traoré, P., Ahipo, Y., Louste, C., 2009. A robust and efficient finite volume 856 scheme for the discretization of diffusive flux on extremely skewed meshes 857 in complex geometries. Journal of Computational Physics 228, 5148-5159. 858 doi:10.1016/j.jcp.2009.04.007. 859
- Trias, F.X., Gorobets, A., Oliva, A., 2015. Turbulent flow around a square 860 cylinder at reynolds number 22000: a dns study. Computer and Fluids 123, 861 87-98. doi:10.1016/j.compfluid.2015.09.013. 862
- Vickery, B., 1966. Fluctuating lift and drag on a long cylinder of square cross 863 section in a smooth and turbulent flow. Journal of Fluid Mechanics 25, 481– 864 494. doi:10.1017/S002211206600020X. 865
- Wang, G., Vanka, S.P., 1997. Les of flow over a square cylinder, in: Chollet, 866 J.P., Voke, P.R., Kleiser, L. (Eds.), Direct and Large-Eddy Simulation II, pp. 867 397-400. doi:10.1007/978-94-011-5624-0_37.

868

- Wilkinson, R., 1974. On the vortex-induced loading on long bluff cylinders. 869 Ph.D. thesis. 870
- Yakhot, V., Orszag, S., Thangam, S., Gatski, T., Speziale, G., 1992. Develop-871 ment of turbulence models for shear flows by a double expansion technique. 872 Physics of Fluids A4, 1510–1520. doi:10.1063/1.858424. 873
- Younis, B.A., Przulj, V.P., 2005. Computation of turbulent vortex shedding. 874
- Computational Mechanics 37, 408-425. doi:10.1007/s00466-005-0713-2. 875

- ⁸⁷⁶ Yu, D., Kareem, A., 1996. Two dimensional simulation of flow around rectan-
- $_{\tt 877}$ $\,$ gular prisms. Journal of Wind Engeneering and Industrial Aerodynamics ,
- ⁸⁷⁸ 131–161doi:10.1016/s0167-6105(96)00076-1.
- ⁸⁷⁹ Zhang, Z., Xu, F., 2020. Spanwise length and mesh resolution effects on simu-
- lated flow around a 5:1 rectangular cylinder. Journal of Wind Engineering &
- Industrial Aerodynamics 202, 104186. doi:10.1016/j.jweia.2020.104186.
- ⁸⁸² Zhu, G., Huang, S., Li, Q., 2020. Large-eddy simulation of the inflow turbulence
- transport and aerodynamics of a rectangular 5:1 cylinder with high-order
 numerical methods. Journal of Wind Engineering & Industrial Aerodynamics
- ⁸⁸⁵ 207, 104366. doi:10.1016/j.jweia.2020.104366.