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Progressive Damage Analysis of Composite Structures Using Higher-Order Layer-Wise Elements

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Abstract

The objective of the current work is the development of a numerical framework for the simulation of damage in composite structures using explicit time integration. The progressive damage is described using a Continuum Damage Mechanics (CDM) based material model, CODAM2, in which the damage initiation and progression are modelled using Hashin's failure criteria and crack-band theory, respectively. The structural modelling uses higher-order theories based on the Carrera Unified Formulation (CUF). The current work considers 2D-CUF models where Lagrange polynomials are used to represent the displacement field through the thickness of each ply, resulting in a layer-wise element model. Numerical assessments are performed on coupon-level specimens, and the results are shown to be in good agreement with reference numerical predictions and experimental data, thus verifying the current implementation for progressive tensile damage. The capability of the proposed framework in increasing the polynomial expansion order through the ply thickness, and its influence on the global behaviour of the structure in the damaged state, is demonstrated. The advantages of using higher-order structural models in achieving significant improvements in computational efficiency are highlighted.

Keywords: CODAM2, CUF, explicit damage analysis, higher-order structural modelling

1 Introduction

Fibre reinforced composites have become very popular as an engineering material system in recent decades due to their desirable properties. This is especially true in the aerospace industry due to the high specific stiffness and strength of such materials. However, wide-spread adoption of composites has still not been achieved, due to uncertainties regarding their non-linear behaviour. Laminated composites typically exhibit extremely complex failure modes, and the accurate modelling of such mechanisms is a challenging task. Nevertheless, several attempts have been made by researchers to develop damage models for composite materials [1, 2].

Computational damage models for composite structures can be generalised into two broad categories. The first approach, based on discrete modelling, involves the explicit geometrical representation of cracks within the structure. Such a technique results in a physically realistic description of the damage mechanisms and their interactions, but at the expense of greatly increased computational effort. The discrete modelling approach typically makes use of interface elements, based on cohesive zone modelling, to simulate both matrix cracks within the ply as well as delamination between the plies [3–7]. Other discrete modelling techniques involve the eXtended-Finite Element Method (XFEM) where enriched kinematics are used to describe the displacement discontinuity across the crack [8, 9], and the floating node method [10, 11].

An alternative approach to discrete damage modelling is based on the concept of continuum damage mechanics (CDM), where the continuity of the displacement field in the finite element mesh is maintained. In contrast to discrete damage modelling, the individual cracks within the composite material are smeared out in CDM approaches, and replaced with damage parameters in the constitutive relationship to describe the influence of such cracks on the global structural behaviour. CDM techniques are popular due to their ease of implementation and relatively low computational cost. However, they generally exhibit a strong mesh dependency, which is reduced by scaling the fracture energy using a characteristic element length, as described by the crack-band theory [12]. Some early works on continuum damage modelling of composite laminates include the works of Ladaveze et al. [13] and Matzenmiller et al. [14]. Some recent works based on continuum damage modelling include the investigation of size effects in notched composites [15], impact analysis of composite plates [16], and progressive damage analysis of composite pressure vessels [17]. A popular approach for composite damage modelling is a combination of CDM to describe intralaminar damage within the ply, and discrete approaches such as the cohesive zone method to model delamination. Such an approach constitutes a good compromise between computational effort and accuracy. Some examples of the application of the combined technique are the failure analysis of open-hole tension laminates [18], damage and delamination analysis of hybrid composite joints [19], and impact analysis of composites [20, 21]. More recently, the phase-field approach has emerged as a promising technique to tackle various damage mechanisms [22]. The current work considers a purely continuum damage approach, where intralaminar damage within the ply is described using the COmposite DAMage (CODAM) model. The CODAM model was originally developed as a sub-laminate based continuum damage model [23], and applied to the simulation of braided composite tubes under axial crushing [24]. CODAM has

also been combined with an adaptive local cohesive zone method for the efficient analysis of composites under axial crushing and transverse impact loading [25]. The second-generation damage model, termed CODAM2, is a strain-based damage formulation at the ply level which can be applied to the macro and meso-scale [26–28]. This version is implemented as a built-in material model (MAT219) in the explicit finite element software LS-DYNA. The latest extension of CODAM2 includes a stress-based criterion for damage initiation and a coupling between matrix damage and delamination [29], and is the version that has been implemented in the current work. The structural modelling is performed using higher-order structural theories based on the Carrera Unified Formulation (CUF) [30], which is a generalised framework to develop refined 1D and 2D models where the kinematic field is enriched via the use of cross-section and thickness expansion functions, respectively. Such an approach results in 3D-like accuracy of the solution while avoiding the computational costs associated with standard 3D-FEA. The current work aims to extend the capabilities of CUF as a virtual testing framework. The numerical results shown in this paper stem from the implementation of CODAM2 within an in-house FE code based on CUF and referred to as MUL2. Among other advantages, the in-house code allows for the full control of the structural modeling, including the complete access to all the FE arrays. To the best of our knowledge, the literature on the combined use of damage models as CODAM2 and refined structural models as CUF is very limited. Such scarcity augments if layer-wise models are considered. Thus, the present paper can provide insights on damage modeling capabilities combined with structural modeling options not available in commercial codes. Previous works in CUF include the nonlinear analysis of thin-walled structures [31, 32], micromechanical progressive failure analysis of composites [33], and multi-scale analysis [34, 35]. This paper is divided into the following parts - Section 2 describes the development of 2D structural theories in CUF and Section 3 provides a brief overview of the CODAM2 intralaminar damage model. Some numerical test cases are given in Section 4, and the results are discussed in Section 5. The conclusions of the present work are highlighted in Section 6.

2 Structural theories and FE formulation

2.1 Carrera Unified Formulation

Consider a 3D physical structure which consists of a laminated plate, as shown in Fig. 1. The 2D-CUF model is also schematically shown in the figure, with the in-plane geometry of the structure oriented along the x-y plane, and the thickness described along the z-axis. The displacement field is defined in CUF as

$$u(x, y, z) = F_0(z)u_0(x, y) + F_1(z)u_1(x, y) + \ldots + F_M(z)u_N(x, y)$$

$$v(x, y, z) = F_0(z)v_0(x, y) + F_1(z)v_1(x, y) + \ldots + F_M(z)v_N(x, y)$$

$$w(x, y, z) = F_0(z)w_0(x, y) + F_1(z)w_1(x, y) + \ldots + F_M(z)w_N(x, y)$$
(1)

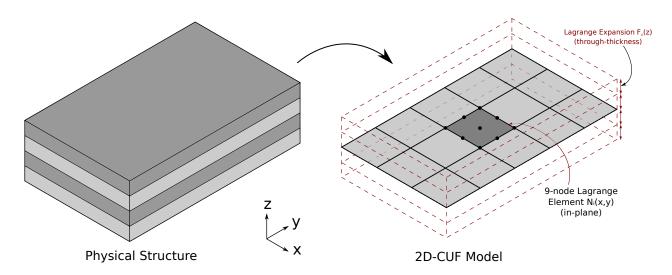


Figure 1: Layer-wise modelling of composite laminates in CUF. This approach uses 9-node second-order quadrilateral (Q9) elements to model the in-plane geometry, while Lagrange polynomial expansion functions are used for the explicit description of individual plies.

F are expansion functions of the thickness coordinate and define the structural theory adopted. Considering a layer-wise (LW) modelling approach, the displacements can be written in a compact form as

$$\mathbf{u}^{k}(x,y,\zeta_{k}) = F^{k}_{\tau}(\zeta_{k})\mathbf{u}^{k}_{\tau}(x,y), \tau = 0, 1, \dots M$$

$$\tag{2}$$

where k is the ply index of the laminated plate, the expansion function $F_{\tau}(\zeta_k)$ describes the kinematics through the thickness of ply k with the thickness domain $\zeta_k \in [-1, 1]$, and $\mathbf{u}_{\tau}^k(x, y)$ are the generalised displacements. The number of terms in the expansion function is denoted by M. The choice of the expansion, F_{τ} , and the number of terms, M, determines the structural theory used in the analysis. The current work considers Lagrange interpolation polynomials to define the expansion through the thickness. In CUF terminology, such a choice is termed the Lagrange-Expansion (LE) class. The current work considers first-order linear expansions (LE1), second-order quadratic expansions (LE2), and third-order cubic expansions (LE3). The Lagrange polynomial of order N is given by

$$F_{\tau}(\zeta_k) = \prod_{i=0, i \neq s}^{N} \frac{\zeta_k - \zeta_{k_i}}{\zeta_{k_s} - \zeta_{k_i}}$$
(3)

 ζ_{k_s} are located at the prescribed interpolation points. $\zeta_{k_0} = -1$ and $\zeta_{k_N} = 1$ correspond to the top and bottom positions of the k^{th} layer, respectively. The displacements at the layer interfaces obey the following compatibility condition

$$u_{top}^{k} = u_{bottom}^{k+1}, \ k = 1, N_{ply} - 1$$
(4)

The use of thickness functions from the LE class results in purely displacement degrees of freedom, and eliminates rotations. Further details on the use of Lagrange polynomials as a class of expansion function in 1D-CUF models may be found in [36], while an overview of 2D plate modelling in CUF can be found in [37].

2.2 Finite Element Formulation

The stress and strain tensors are defined in vector notation as

$$\boldsymbol{\sigma} = \{\sigma_{xx}, \sigma_{yy}, \sigma_{zz}, \sigma_{xy}, \sigma_{xz}, \sigma_{yz}\}$$

$$\boldsymbol{\varepsilon} = \{\varepsilon_{xx}, \varepsilon_{yy}, \varepsilon_{zz}, \varepsilon_{xy}, \varepsilon_{xz}, \varepsilon_{yz}\}$$
(5)

The linear strain-displacement relation is then expressed as

$$\boldsymbol{\varepsilon} = \mathbf{B}\mathbf{u}$$
 (6)

where the differential operator, \mathbf{B} , is

$$\mathbf{B} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 & 0\\ 0 & \frac{\partial}{\partial y} & 0\\ 0 & 0 & \frac{\partial}{\partial z}\\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} & 0\\ \frac{\partial}{\partial z} & 0 & \frac{\partial}{\partial x}\\ 0 & \frac{\partial}{\partial z} & \frac{\partial}{\partial y} \end{bmatrix}$$

The structural framework adopted in this paper can handle nonlinear geometrical relations [31], however, in the current study, it was deemed sufficient to limit our analysis to linear geometries in order to ease the verification of results and investigate the effect of higher-order theories on damage onset and propagation. The constitutive relation is given by

$$\boldsymbol{\sigma} = \mathbf{C}^{sec} \boldsymbol{\varepsilon} \tag{7}$$

where \mathbf{C}^{sec} is the secant material stiffness matrix, obtained from the CODAM2 damage model as described in Section 3. The use of the secant stiffness matrix has several advantages and disadvantages as well as numerical peculiarities. For a detailed analysis and guidelines on the use of this matrix the reader may refer to [14, 29]. The damaged stress state is then represented by $\boldsymbol{\sigma}$. The in-plane geometry is discretised with 4- or 9-node quadrilateral finite elements (Q4 or Q9), using nodal interpolation functions $N_i(x, y)$, leading to the following 3D form of the displacement field

$$\mathbf{u}(x, y, z) = N_i(x, y) F_\tau(z) \mathbf{u}_{\tau i} \tag{8}$$

2.3 Explicit time integration

As commonly implemented in commercial codes for this class of problems, an explicit scheme was adopted for the numerical solution. The semi-discrete balance of momentum is given by

$$\mathbf{M\ddot{a}}^{t+\Delta t} = \mathbf{f}_{ext}^{t+\Delta t} - \mathbf{f}_{int}^{t+\Delta t} \tag{9}$$

where **M** is the mass matrix and \mathbf{f}_{ext} and \mathbf{f}_{int} are the external and internal force vectors, respectively. The internal force vector can be calculated via the integration of stress as shown in Table 1. The nonlinear dynamic problem is solved explicitly using the central difference scheme, whose formulation can be found, for instance, in [38]. The version of the central difference scheme employed in the current work approximates the velocity at the mid-interval such that

$$\dot{\mathbf{u}}^{t+\frac{1}{2}\Delta t} = \frac{\mathbf{u}^{t+\Delta t} - \mathbf{u}^t}{\Delta t} \tag{10}$$

where \mathbf{u} and $\dot{\mathbf{u}}$ are the displacement and velocity vectors, respectively. Equation 10 is re-written to obtain an expression for the displacement update

$$\mathbf{u}^{t+\Delta t} = \mathbf{u}^t + \Delta t \ \dot{\mathbf{u}}^{t+\frac{1}{2}\Delta t} \tag{11}$$

The updated displacements are used to calculate the new strain and stress states, leading to the computation of $\mathbf{f}_{int}^{t+\Delta t}$. The updated acceleration is directly computed from Eq. 9 as

$$\ddot{\mathbf{u}}^{t+\Delta t} = \mathbf{M}^{-1} \{ \mathbf{f}_{ext}^{t+\Delta t} - \mathbf{f}_{int}^{t+\Delta t} \}$$
(12)

The mid-interval velocity for the next cycle, required for Eq. 11, is computed from the acceleration as

$$\dot{\mathbf{u}}^{t+\frac{3}{2}\Delta t} = \dot{\mathbf{u}}^{t+\frac{1}{2}\Delta t} + \Delta t \ \ddot{\mathbf{u}}^{t+\Delta t} \tag{13}$$

At the start of the solution (t = 0), the mid-interval velocity $\dot{\mathbf{u}}^{\frac{1}{2}\Delta t}$ is required to solve Eq. 11. This term is determined by the following assumption

$$\dot{\mathbf{u}}^{\frac{1}{2}\Delta t} = \dot{\mathbf{u}}^0 + \frac{1}{2}\Delta t \ \ddot{\mathbf{u}}^0 \tag{14}$$

where the initial velocity, $\dot{\mathbf{u}}^0$, and the initial acceleration, $\ddot{\mathbf{u}}^0$, are based on the initial conditions of the system. Finally, a lumped mass matrix is considered in Eq. 12, which results in computationally inexpensive vector multiplication. The row summing technique is utilised to obtain the lumped mass matrix. The algorithm for the central difference scheme with mid-interval velocities has been summarised in Table 1.

The critical time step for the analysis is determined by computing the highest frequency of the system,

Table 1: Algorithm for the central difference time integration scheme [38]

Initialise \mathbf{u}^0 and $\dot{\mathbf{u}}^0$ (t = 0) Compute the lumped mass matrix **M** Compute initial mid-interval velocity: $\dot{\mathbf{u}}^{\frac{1}{2}\Delta t} = \dot{\mathbf{u}}^0 + \frac{1}{2}\Delta t \ddot{\mathbf{u}}^0$ **Loop over the time increments For each i-th time step** $(\mathbf{t}_i = \mathbf{t}_{i-1} + \Delta \mathbf{t})^*$: 1. Compute new displacements: $\mathbf{u}^{t+\Delta t} = \mathbf{u}^t + \Delta t \dot{\mathbf{u}}^{t+\frac{1}{2}\Delta t}$ 2. Compute displacement increment: $\Delta \mathbf{u} = \mathbf{u}^{t+\Delta t} - \mathbf{u}^t$ 3. For each integration point: \rightarrow Compute updated strains: $\varepsilon^{t+\Delta t} = \varepsilon^t + \mathbf{B}\Delta \mathbf{u}$ \rightarrow Compute updated stress: $\sigma^{t+\Delta t} = \mathbf{C}^{sec}\varepsilon^{t+1}$ 4. Compute internal force vector: $\mathbf{f}_{int}^{t+\Delta t} = \int_V \mathbf{B}^T \sigma^{t+\Delta t} dV$ 5. Compute new accelerations: $\ddot{\mathbf{u}}^{t+\Delta t} = \mathbf{M}^{-1}\{\mathbf{f}_{ext}^{t+\Delta t} - \mathbf{f}_{int}^{t+\Delta t}\}$ 6. Compute new mid-interval velocities: $\dot{\mathbf{u}}^{t+\frac{3}{2}\Delta t} = \dot{\mathbf{u}}^{t+\frac{1}{2}\Delta t} + \Delta t \ddot{\mathbf{u}}^{t+\Delta t}$ *For the sake of clarity, subscripts are reported only here

 ω_{max} , and using the following expression

$$\Delta t_{critical} = \frac{2}{\omega_{max}} \tag{15}$$

The highest frequency may be calculated via power iteration schemes as in [39].

3 CODAM2 intralaminar damage model

The current work deals with fiber-dominated tensile progressive damage via the ply-based form of the CODAM2 stress-based damage model has been considered in the current work. Damage initiation occurs when the damage initiation function $F_{\alpha} \ge 1$, in the fibre ($\alpha = 1$) and transverse ($\alpha = 2$) direction. Damage initiation along the longitudinal direction, i.e. fibre damage, is described by F_1 , which is a maximum stress criterion given by

$$F_1 = \frac{\sigma_{11}}{X_T} \tag{16}$$

where X_T is the fibre tensile strength. Stress and strain components with the subscripts $\{11, 22, 12\}$ indicate the fields that have been rotated into the material reference system. Similarly, damage initiation along the transverse direction, i.e. matrix damage, is described by Hashin's quadratic failure criterion [40] as

$$F_2 = \left(\frac{\sigma_{22}}{Y_T}\right)^2 + \left(\frac{\tau_{12}}{S_L}\right)^2 \tag{17}$$

where Y_T and S_L are the transverse tensile and in-plane shear strength, respectively. The damage progression criteria requires the equivalent strain measures ε_1^{eq} and ε_2^{eq} in the longitudinal and transverse directions respectively, as damage drivers, and are defined as

$$\varepsilon_1^{eq} = |\varepsilon_{11}| \tag{18}$$

$$\varepsilon_2^{eq} = \sqrt{(\gamma_{12})^2 + (\varepsilon_{22})^2}$$
 (19)

where γ_{12} refers to the shear strain. In this paper, however, the shear nonlinearity was not considered. The present paper is on the correct implementation of CODAM2 considering fiber-dominated test cases. Therefore matrix-driven effects such as nonlinear shear, through-thickness stresses, and delamination only play a minor role. The work-conjugate equivalent stress measures are

$$\sigma_1^{eq} = \sigma_{11} \tag{20}$$

$$\sigma_2^{eq} = \frac{\tau_{12}\gamma_{12} + \sigma_{22}\varepsilon_{22}}{\sqrt{(\gamma_{12})^2 + (\varepsilon_{22})^2}}$$
(21)

The strains at damage initiation are given by

$$\varepsilon_{\alpha}^{i} = \varepsilon_{\alpha}^{eq}|_{F_{\alpha}=1}, \quad \alpha = 1, 2 \tag{22}$$

Subsequently, the strains at damage saturation are defined as

$$\varepsilon_1^s = \frac{2g_1^f}{X_T} \quad and \quad \varepsilon_2^s = \frac{2g_2^f}{T} \tag{23}$$

where g_{α}^{f} is the fracture energy density, and $T = \sigma_{2}^{eq}|_{F_{2}=1}$ is the peak value of the equivalent transverse stress σ_{2}^{eq} . The crack-band approach [12] is used to reduce mesh dependency by scaling the experimentally determined fracture energy G_{α}^{f} , using a characteristic length parameter of the element, as follows

$$g^f_{\alpha} = \frac{G^f_{\alpha}}{l^*}, \quad \alpha = 1,2 \tag{24}$$

where l^* is the characteristic element length. The current work considers $l^* = (V_{GP})^{\frac{1}{3}}$, where V_{GP} is the Gauss point volume of the given element. Such a choice for the characteristic length is consistent with other works on CUF for progressive damage and proved to be robust in ensuring mesh independency [33]. The Gauss point volume is the share of the element volume at each Gauss point; that is, in the case of one Gauss point, this volume coincides with the element volume, with two Gauss points, each volume is half of the element volume. In contrast to [27], the local form of the CODAM2 model is implemented herein, such that Eq. 24 is applicable for both longitudinal and transverse directions. The damage parameters ω_{α} , used to evaluate damage progression, are defined as

$$\omega_{\alpha} = \left(\frac{\langle \varepsilon_{\alpha}^{eq} - \varepsilon_{\alpha}^{i} \rangle}{\varepsilon_{\alpha}^{s} - \varepsilon_{\alpha}^{i}}\right) \left(\frac{\varepsilon_{\alpha}^{s}}{\varepsilon_{\alpha}^{eq}}\right), \quad \alpha = 1, 2$$
(25)

where $\langle \cdot \rangle$ denotes the Macaulay bracket. This degradation model leads to bilinear laws as detailed in [27] in which comparisons with other forms of degradation were carried out. Using the damage variables, the 3D form of the secant stiffness matrix in the damaged state [29] is written as

where $\Delta = 1 - R_2 \nu_{23} \nu_{32} - R_1 R_2 \nu_{12} \nu_{21} - 2R_1 R_2 \nu_{31} \nu_{12} \nu_{23} - R_1 \nu_{31} \nu_{13}$ and R_{α} denotes the stiffness reduction factor, given by

$$R_{\alpha} = (1 - \omega_{\alpha}), \quad \alpha = 1, 2 \tag{27}$$

Such formulation considers the thermodynamic consistency as detailed in [14, 28, 29]. Also, in keeping with the formulation in [29], the out-of-plane shear moduli are not degraded as cohesive elements are usually employed to capture the through-thickness damage (delamination). In this work, such elements were not implemented as the focus is on test cases that involve fibre dominated damage. Since the current structural formulation can accommodate interlaminar stress analysis [41], future studies will be carried out to account for delamination. Finally, the stress state is computed as

$$\boldsymbol{\sigma} = \mathbf{C}^{dam} \boldsymbol{\varepsilon} \tag{28}$$

4 Numerical Examples

The current section consists of a series of numerical assessments which serve as validation cases for the proposed modelling approach. The material system used in each case is IM7/8552 carbon fibre reinforced polymer (CFRP) with ply thickness of 0.125 mm. Its properties are given in Table 2.

4.1 Single element tests

The first set of numerical assessments consists of single element tests under uni-axial strain conditions, which are a convenient way of verifying the implementation of the damage model, since the different failure modes

Material	E_1 [GPa]	E_2 [GPa]	E_3 [GPa]	G_{12} [GPa]	G_{13} [GPa]	G_{23} [GPa]	ν_{12}	ν_{13}	ν_{23}
IM7/8552 CFRP	165.0	9.0	9.0	5.6	5.6	2.8	0.34	0.34	0.5
		X_T [MPa]	Y_T [MPa]	S_L [MPa]	$G_1^f \; [kJ/m^2] \; [43]$	$G_2^f \; [kJ/m^2] \; [29]$	$ ho \; [Kg/m^3]$		
		2560.0	73.0	90.0	120.0	2.6	1700.0		

Table 2: Material properties of the IM7/8552 CFRP material system [42]

can be independently evaluated. In each case, the in-plane geometry is modelled using a 4-node quadrilateral element (Q4), with the ply thickness modelled using a linear Lagrange polynomial (LE1). The use of the single element test is typical for this class of problems, and it is necessary to verify the implementation. The analysis is performed on a Q4 element to be as close as possible to the structural modeling adopted in the referenced papers. The first test involves a single element under longitudinal tension, which results in a fibre failure mode. The stress-strain curve for this case is shown in Fig. 2a. Next, the single element is subjected to transverse tension, resulting in a matrix failure mode. The stress-strain curve for this case has been plotted in Fig. 2b. The final assessment is the tensile loading of a single element consisting of a quasi-isotropic laminate with a $[90/45/0/ - 45]_{2s}$ layup. For this case, the predicted stress-strain curve is shown in Fig. 3. The results are compared to the CODAM2 reference model [27], implemented as a user-defined subroutine in LS-DYNA.

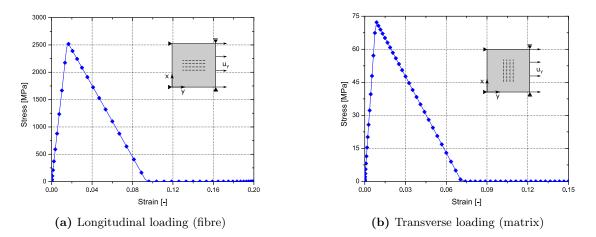


Figure 2: Stress-strain curve of a single element loaded in tension under uni-axial strain conditions. (a) Element loaded in the longitudinal (fibre) direction, and (b) Element loaded in the transverse (matrix dominated) direction

The following observations are made

- 1. The stress-strain curves shown in Fig. 2 follow the bilinear degradation law described by the CODAM2 damage model, and the peak stresses predicted by the CUF models under longitudinal and transverse tension are equal to the fibre and matrix material strengths, respectively, i.e., the analysis output is consistent with the input material strength values.
- 2. The results of the single element laminate obtained by the present implementation is in good agreement with reference numerical results, as shown in Fig. 3.

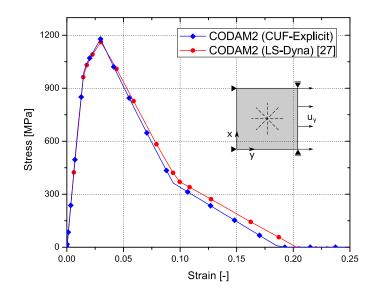


Figure 3: Stress-strain response of the single element $[90/45/0/-45]_{2s}$ IM7/8552 CFRP quasi-isotropic laminate in tension under uniaxial strain condition

The above observations verify the implementation of the CODAM2 damage model in CUF-Explicit.

4.2 Centre-notched tension specimen

The current example concerns the analysis of a centre-notched tensile (CNT) specimen. The structure is shown schematically in Fig. 4, and the ply stacking sequence is $[45/90/-45/0]_{4s}$. The coupon is constrained at one end, and a displacement u_y is applied to the opposite end. Various scales of the coupon have been numerically analysed, in order to demonstrate the capability of the current framework in predicting size-effects in composite structures. The various scales used, and their dimensions, are listed in Table 3. The current example is based on the works of [27], which provides reference numerical results based on the LS-DYNA implementation of the CODAM2 model. In addition, reference numerical results are also obtained from the ABQ_DLR model, which is a Ladavèze-based damage model, implemented as a user-material (VUMAT) in ABAQUS/Explicit [27]. The peak strengths predicted in the current work have also been compared with experimentally obtained data [44]. The mesh used in the CUF-Explicit analyses (scales 1-16) consist of 132 quadratic (see Fig. 5, second-order, Q9) elements within the plane, while the scale-24 analysis consists of a 244 Q9 mesh. In each case, the element distribution was kept the same with the size of the element edges scaling as the dimensions of the specimen. The mesh-size in the fracture process zone is approximately $4 \ mm \ \times 0.5 \ mm$ for the case of the scale-8 mesh. All the CUF models use a linear (first-order, LE1) Lagrange polynomial expansion to explicitly model each ply. In the case of the scale-8 specimen, three models are considered, where the plies are modelled using a linear (first-order, LE1), quadratic (second-order, LE2) and cubic (third-order, LE3) expansion, respectively, as described in Section 2.1. Also, various Q9 meshes were used by refining in the vicinity of the crack tip.

Figure 6 shows the stress-strain curve for the scale-8 CNT specimen, as obtained from the various modelling

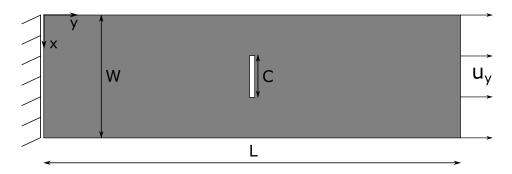


Figure 4: Schematic representation of the centre-notched tensile specimen geometry and applied boundary conditions

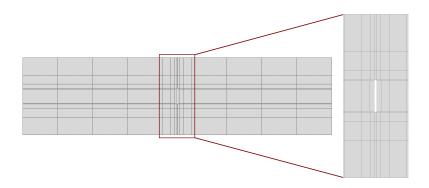


Figure 5: FE mesh with 132 Q9 elements

Scale	Notch Length C [mm]	Specimen Width [mm]	Specimen Length [mm]
1	3.2	15.9	63.5
2	6.4	31.8	127.0
4	12.7	63.5	254.0
8	25.4	127.0	508.0
16	50.8	254.0	508.0
24*	76.2	381.0	1016.0

Table 3: Dimensions of the various scales of the CNT specimen

* Virtual test sample

approaches and meshes. The bold horizontal lines indicate the minimum and maximum strength values obtained in experiments. The peak stress (strength) for the various scales have been plotted in Fig. 7. Comparisons have been made with experimental and reference numerical results in both cases. The experimental results were retrieved from [44] in which error bars are available. The influence of the structural dimensions on the overall computational cost of the analysis has been investigated, in terms of the degrees of freedom (DOF) required, as shown in Fig. 8a, and normalised computational time, as shown in Fig. 8b. The computational times have been normalised for each model type using the computational time required for the analysis of the scale-1 specimen.

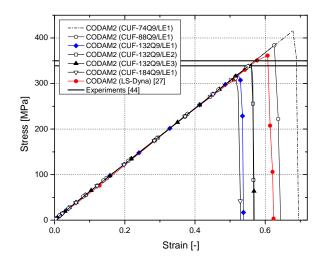


Figure 6: Stress-strain curve for the scale-8 centre-notched tension specimen with $[45/90/-45/0]_{4s}$ layup. The horizontal bold lines indicate minimum and maximum strength values obtained in experiments

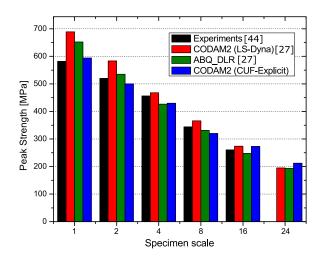


Figure 7: Comparison of the peak strength obtained by various numerical approaches and experimental measurements for the centre-notched tensile specimens with $[45/90/-45/0]_{4s}$ layup. The influence of specimen size on the peak strength of the specimen is clearly observed.

Based on the results, the following comments can be made

1. From Fig. 6, a brittle behaviour of the specimen can be observed, with a linear elastic increase of the stress until the maximum value is reached, followed by abrupt loss of stiffness leading to failure. The response

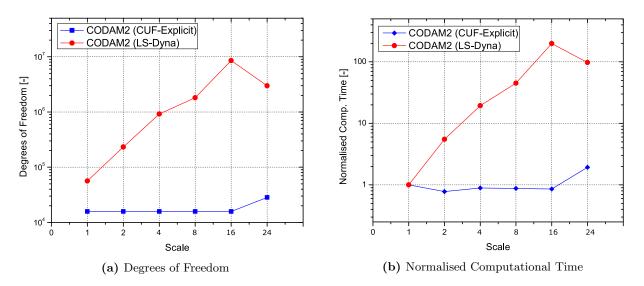


Figure 8: Degrees of freedom and normalised computational time required for the numerical analysis of the various CNT scales in CUF-Explicit and LS-DYNA

predicted by the current approach is in good agreement with reference numerical and experimental results.

- 2. The influence of the ply thickness expansion order, in the case of CUF models, can be seen in Fig. 6. The use of LE1 expansions through the ply results in a lower peak strength compared to LE2, while there is no difference between LE2 and LE3 expansions. The use of LE1 leads to lower accuracies even if finer meshes are used. Such a result is consistent with previous works of the authors on static analyses [30].
- 3. A clear size-effect can be observed in Fig. 7, based on the peak strengths, through the scales of the CNT specimen. The predictions of the current framework are in good agreement with reference numerical and experimental results.
- 4. The scale-24 analysis (for a fictitious large scale structure where no experimental results are available) is performed using a coarser mesh density in LS-DYNA, resulting in a reduction of the computational effort. The CUF model requires a finer mesh compared to the other scales, but nevertheless maintains significant computational efficiency compared to LS-DYNA.

4.3 Over-height compact tension test of dispersed-ply laminate

The last numerical assessment considers an over-height compact tension (OCT) specimen with a dispersed ply sequence of $[90/45/0/ - 45]_{4s}$, resulting in a quasi-isotropic laminate. The OCT loading geometry results in a stable crack growth, while the use of a dispersed ply sequence makes the laminate less prone to delamination. A schematic representation of the OCT with dimensions and applied boundary conditions is given in Fig. 9. A gradually increasing displacement u (up to 1.0 mm) is prescribed on each pin in opposite directions, leading to a pin opening displacement (POD) of 2u. The CUF-Explicit analysis uses a mesh consisting of 392 quadratic (Q9) elements within the plane, see Fig. 10, and three models are considered with linear (LE1), quadratic

(LE2) and cubic (LE3) ply thickness expansions, respectively. Reference numerical results are obtained from [27], while experimental data is available in [45].

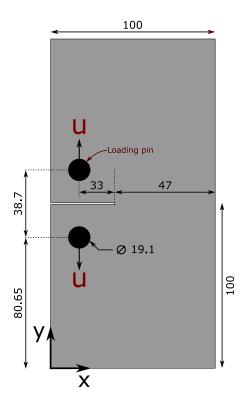


Figure 9: Schematic representation of the over-height compact tensile specimen with a $[90/45/0/-45]_{4s}$ quasiisotropic dispersed ply sequence along with loading conditions (dimensions in mm)

The predicted and measured load-displacement response of the laminate are overlaid in Fig. 11. The evolution of the crack length as a function of the POD has been plotted in Fig. 12. The crack length is determined by considering the extent of fibre damage saturation in the 0° ply of the laminate, where damage is considered to be saturated when the fibre damage parameter, defined in Eq. 25, reaches a value of unity $(\omega_1 = 1.0)$. The contour plot of fibre and matrix damage in the 0° and 90° plies, respectively, is shown in Fig. 13, for a POD of 1.5 mm.

The following observations are made

1. From Fig. 11, it is seen that the peak force obtained with the LE1 model matches that reported by the reference LS-DYNA solution, while the LE3 model predicts a slightly higher peak force. The post-peak softening curve in both cases is in good general agreement with the experimental curve. The overshoot of the initiation of damage by LE models as compared to experimental results may be due to the absence of delamination modelling in the simulation framework adopted in this paper. In a recent work [41], the same structural modelling was used for delamination problems with a far better correlation with experimental results. LE1 has kinematic features quite similar to those of the other two numerical models from the literature, and this leads to a closer match.

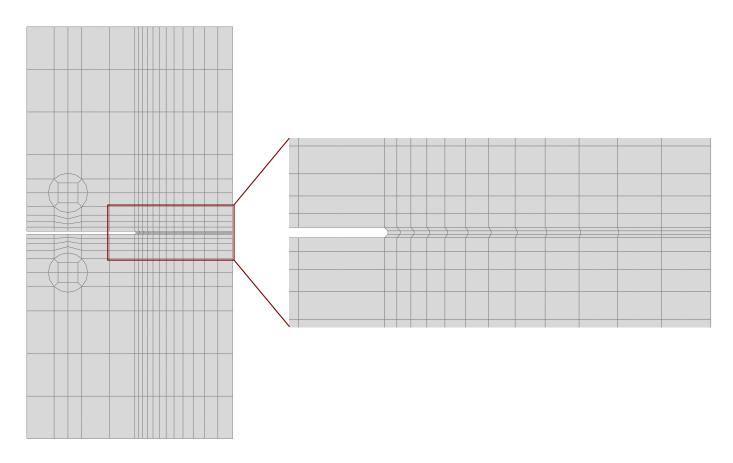


Figure 10: Mesh adopted for the over-height compact tensile specimen with 392 Q9 elements

- 2. The differences between the LS-DYNA and the CUF-Explicit solutions, in the linear regime, stems from the structural theories used in the models. The LS-DYNA model is developed using stacked thick shell elements, while the CUF-Explicit model uses a combination of quadratic (Q9) elements within the plane and Lagrange polynomial expansions of varying order through the thickness for each ply of the laminate.
- 3. A non-negligible amount of numerical oscillations can be observed in the softening curve of the CUF-Explicit models. This stems from the use of a fully-integrated second-order (Q9) in-plane mesh, and the absence of numerical damping.
- 4. The crack length evolution shown in Fig. 12 is in good agreement with experimental data, however with a delayed damage initiation. The same delayed response is also observed in the force-displacement curve in Fig. 11. The numerical models exhibit smooth curves, whereas the experimental one has sudden jumps. As suggested in [45], such a behavior can be attributed to several factors, including the heterogeneous nature of fiber failure in the laminate, manufacturing defects, and interaction of failure mechanisms, including splitting and delamination. The absence of many of these factors in the numerical models may explain such differences.

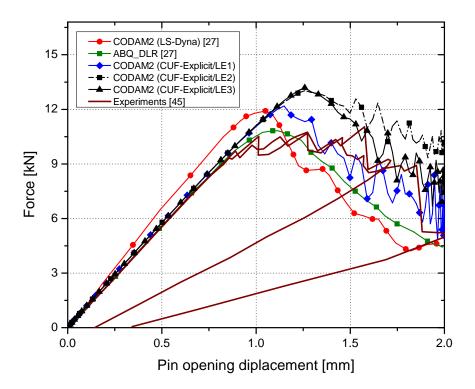


Figure 11: POD-Force curve for the OCT test of dispersed $[90/45/0/-45]_{4s}$ IM7/8552 CFRP laminates

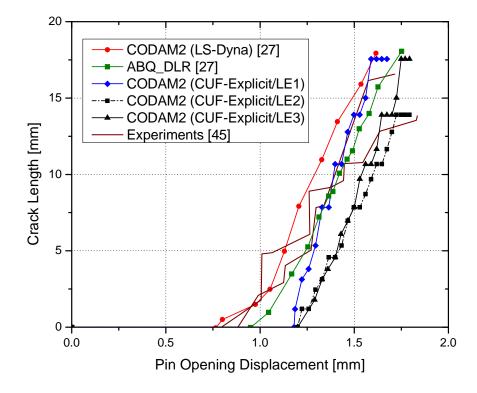
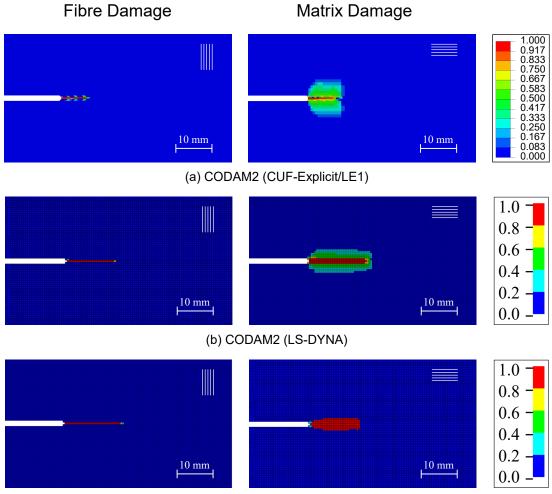


Figure 12: Crack length evolution in the 0° ply for the OCT test of dispersed $[90/45/0/ - 45]_{4s}$ IM7/8552 CFRP laminates



(c) ABQ_DLR

Figure 13: Distribution of the fibre damage in the 0° ply (left) and matrix damage in the 90° ply (right) near the notch for the $[90/45/0/ - 45]_{4s}$ IM7/8552 CFRP quasi-isotropic laminate, at a POD of 1.5 mm. (a) CODAM2 (CUF-Explicit/LE1), (b) CODAM2 (LS-DYNA) [27], and (c) ABQ_DLR [27].

5 Discussion

The initial numerical assessments based on single element tests show the characteristics of the CODAM2 damage model. Tensile loading under uni-axial strain conditions, along and transverse to the fibre direction, results in a bilinear stress-strain behaviour as seen in Fig. 2. Under loading in the fibre direction, the peak stress coincides with the fibre strength X_T following which the load carrying capacity reduces linearly until the damage saturation strain is reached. A similar bilinear behaviour holds for transverse loading where damage occurs in the matrix, with an associated matrix fracture energy $G_2^f = 2.6 \ kJ/m^2$. The peak stress under transverse loading corresponds to the matrix strength Y_T . Furthermore, the area under the curves of Fig. 2a and Fig. 2b are consistent with the input value of the fracture energy for the fibre and matrix, respectively. The stress-strain response of the single element laminate is in good agreement with the reference CODAM2 implementation in LS-DYNA, as seen in Fig. 3. The variation between the two numerical approaches prior to final failure is attributed to the fact that the present work does not consider in-plane shear non-linearity, while it is present in the LS-DYNA implementation.

The use of higher-order finite elements in the continuum damage modelling of composite laminates using explicit codes is largely unexplored in the literature, where the standard approach is to use linear solid elements with reduced integration. Conversely, the present work uses CUF-based higher-order structural models, which can offer several advantages in the progressive damage analysis of composites. The use of expansion functions to enrich the through-thickness kinematics of individual plies results in an accurate resolution of interlaminar stress fields, which are critical inputs to damage models, especially in the case of delamination [41]. The influence of expansion order through the thickness is seen in Fig. 6, where the use of second-order LE2 functions show an improvement in peak stress predictions over the first-order LE1 expansion, keeping the in-plane discretisation constant. Increasing the expansion order further does not lead to any significant effects, since in-plane loading is considered, and delamination is not taken into account. Another characteristic of CUF structural theories is the weak dependency between the size of the structure and the mesh, which is in strong contrast to standard FEA. This is seen in Fig. 7, where the discretisation is kept constant for scales 1-16 of the centre-notched tensile specimen, without affecting the accuracy of the result. Such an approach results in a constant DOF value as seen in Fig. 8a, and consequently, the computational time required also remains fairly constant, as seen in Fig. 8b. On the other hand, an exponential increase in both DOF and computational times can be observed for the case of standard FEA. These aspects could have significant advantages in improving the computational efficiency during the progressive damage analysis of large-scale composite structures.

The final assessment is the simulation of progressive damage in dispersed $[90/45/0/ - 45]_{4s}$ over-height compact tension (OCT) quasi-isotropic laminates, described in Section 4.3. The predictions of the current framework are in good general agreement with reference numerical and experimental data, as seen in Fig. 11 and Fig. 12, which show the global force-displacement response and the crack-length evolution, respectively. The LE3 model predicts a higher peak force compared to the LE1 and the reference LS-DYNA results, but exhibits a good correlation with experimental data in the regime of progressive damage. The predictive capability of the numerical framework can be further improved by a combination of higher-order thickness expansion functions and physically accurate shear behaviour, i.e., considering nonlinear shear effects. A consequence of using higherorder models is the presence of numerical oscillations. This is clearly observed in Fig. 11, where the post peak softening curve shows an oscillatory response. These spurious oscillations can stem from the fact that fullyintegrated higher-order elements are used in the present approach, which can cause excitation of the highest frequency components. The use of different integration schemes, for instance reduced-integration, and damping techniques such as bulk viscosity damping, are possible strategies to mitigate such numerical oscillations, and is an area for future investigation.

6 Conclusion

The current work presents the development of an explicit transient dynamics framework for the progressive tensile damage analysis of composite structures. The ply-level CODAM2 intralaminar damage model, with stress-based failure initiation criteria, has been used to describe the damage behaviour of the composite material, while higher-order structural theories based on the Carrera Unified Formulation are used for the layer-wise structural modelling of the composite laminate. Several validation cases have been considered for the proposed framework, based on the IM7/8552 carbon fibre reinforced polymer material system, and the results have been compared with experimental data as well as reference numerical predictions using LS-DYNA and ABAQUS.

The initial verification of the damage model implementation was demonstrated via a series of single element tests. Coupon-level assessments were then performed on centre-notched tensile laminated specimens with quasi-isotropic layup, over a range of specimen sizes. The final validation case was the over-height compact tension test of quasi-isotropic dispersed laminates. The predictions of the proposed framework were in good general agreement with reference numerical and experimental results. In the case of centre-notched quasi-isotropic dispersed laminates, size-effects were predicted with reasonable accuracy. The advantages of CUF-based higherorder structural models was discussed, in particular the savings in computational costs and the capability to tailor structural models by increasing the order of the thickness expansion functions, according to the requirements of the analysis. These advantages of the proposed framework could thus potentially make it a suitable candidate for the computationally-efficient progressive damage analysis of composite structures. The numerical results look promising for triggering a constructive synergy in terms of computational efficiency and accuracy between CODAM2 and CUF. In particular, the possibility of having complete freedom on the choice of the structural modeling along the thickness looks promising to unleash the modeling capabilities of CODAM2 without prohibitive computational overheads, as in the case of solid elements.

Considering findings from previous papers [46, 47], the use of higher-order terms and LW capabilities is expected to provide decisive advantages in terms of damage onset and propagation in problems with highly localized stress peaks and gradients, e.g., edge effects, or with the presence of tens of layers as in standard structural components. The use of CUF allows us to obtain the full 3D stress field by using the constitutive relations and avoiding post-processing techniques such as the integration of the equilibrium equations. Moreover, the use of 1D or 2D models lead to relaxations of the aspect ratio constraints that 3D models suffer from.

Future investigations include extending the present work to compressive damage, and the inclusion of cohesive zone modelling to account for delamination. A suitable application for the resulting framework would be lowvelocity impact analysis of composite structures.

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